

8/21/03
63/504
Keezo

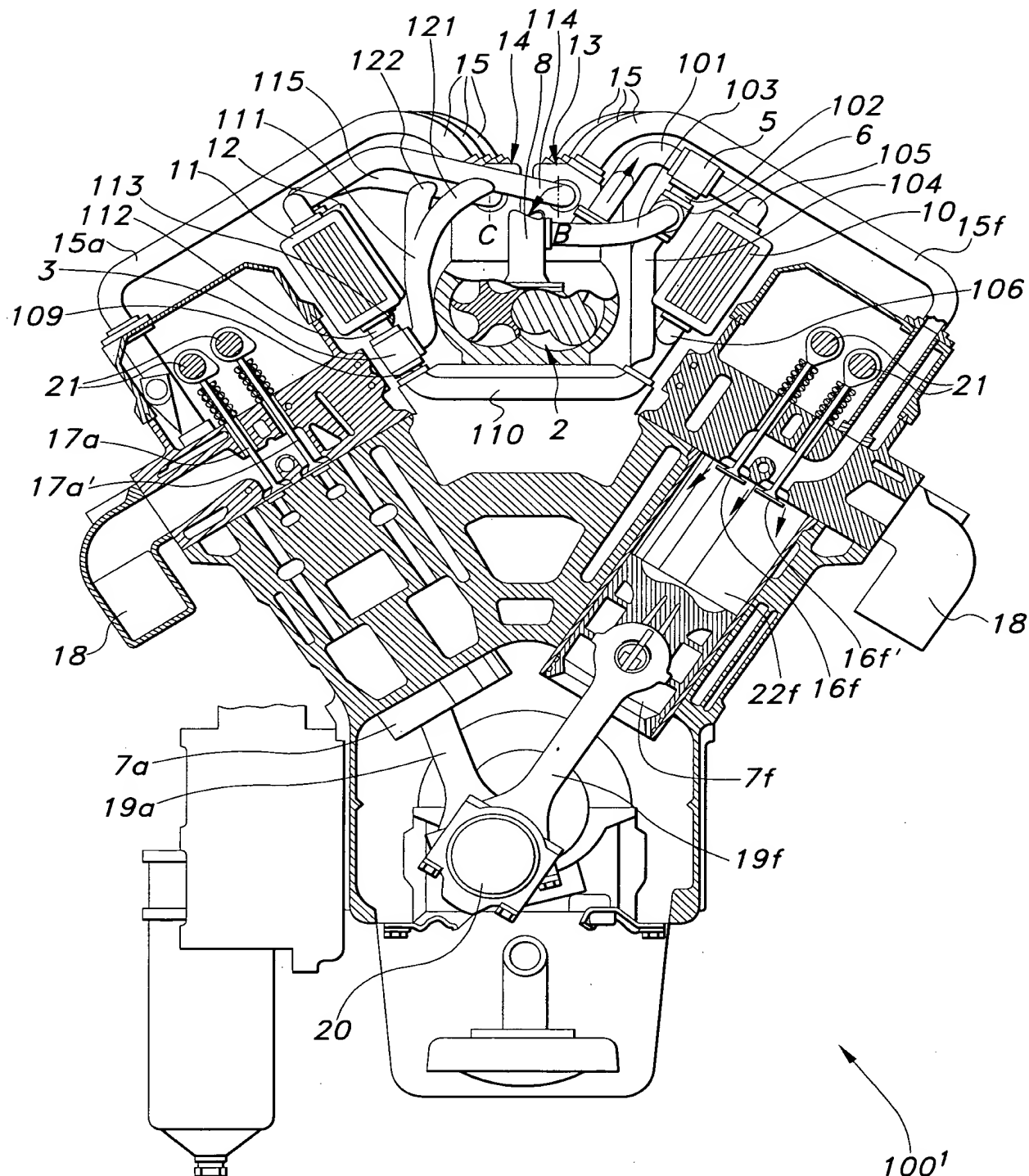


FIG 1

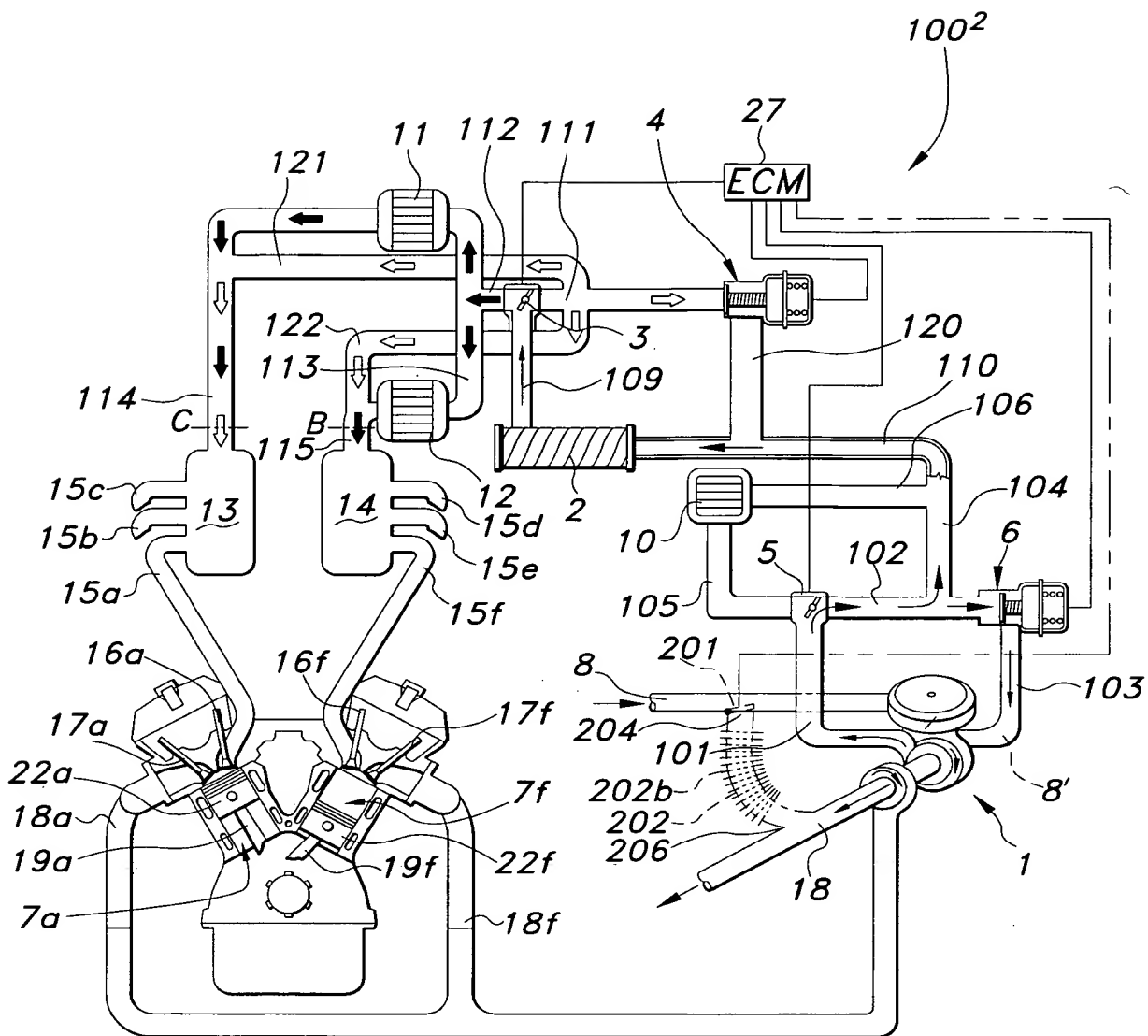


FIG 2

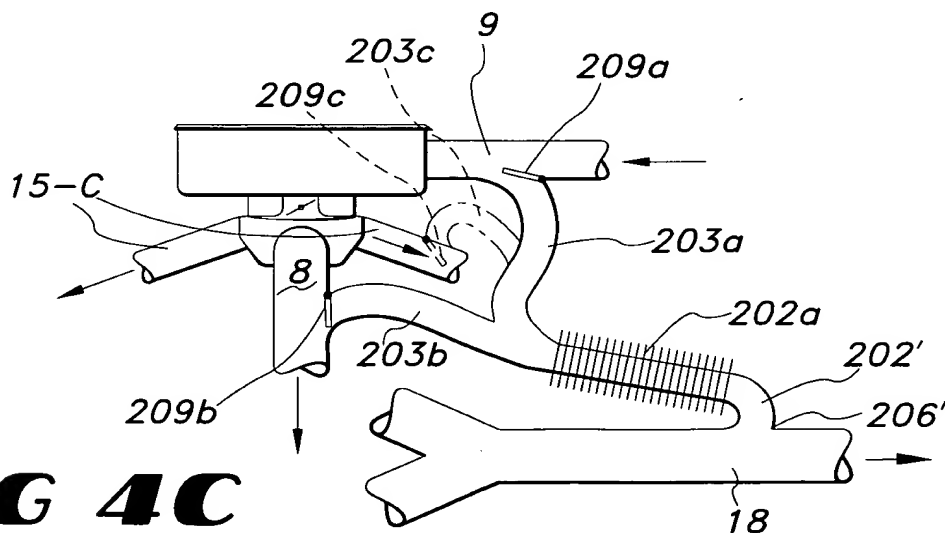


FIG 4C

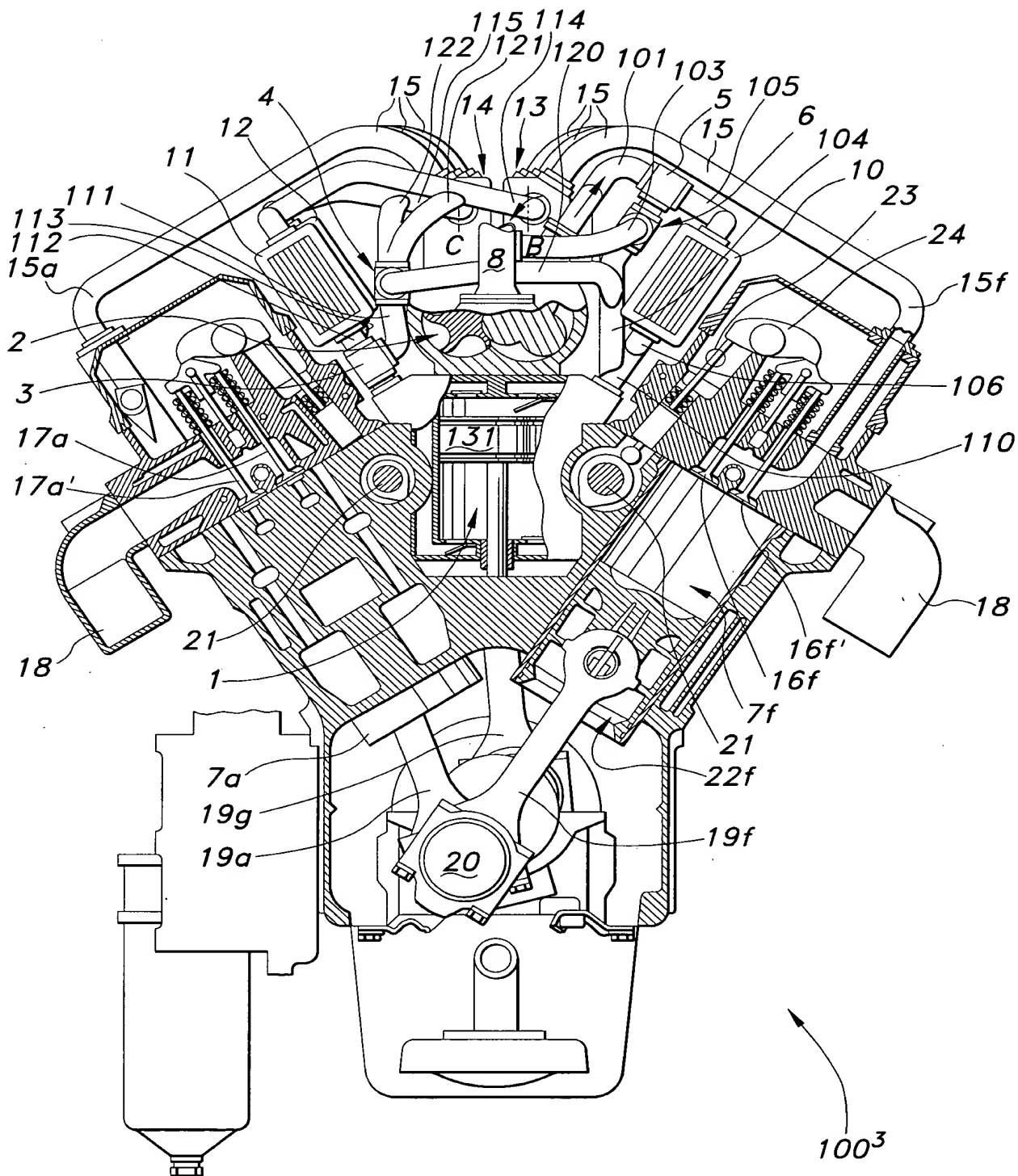


FIG 3

4B
502.1
123
DRAFT
26E250" COTC9980

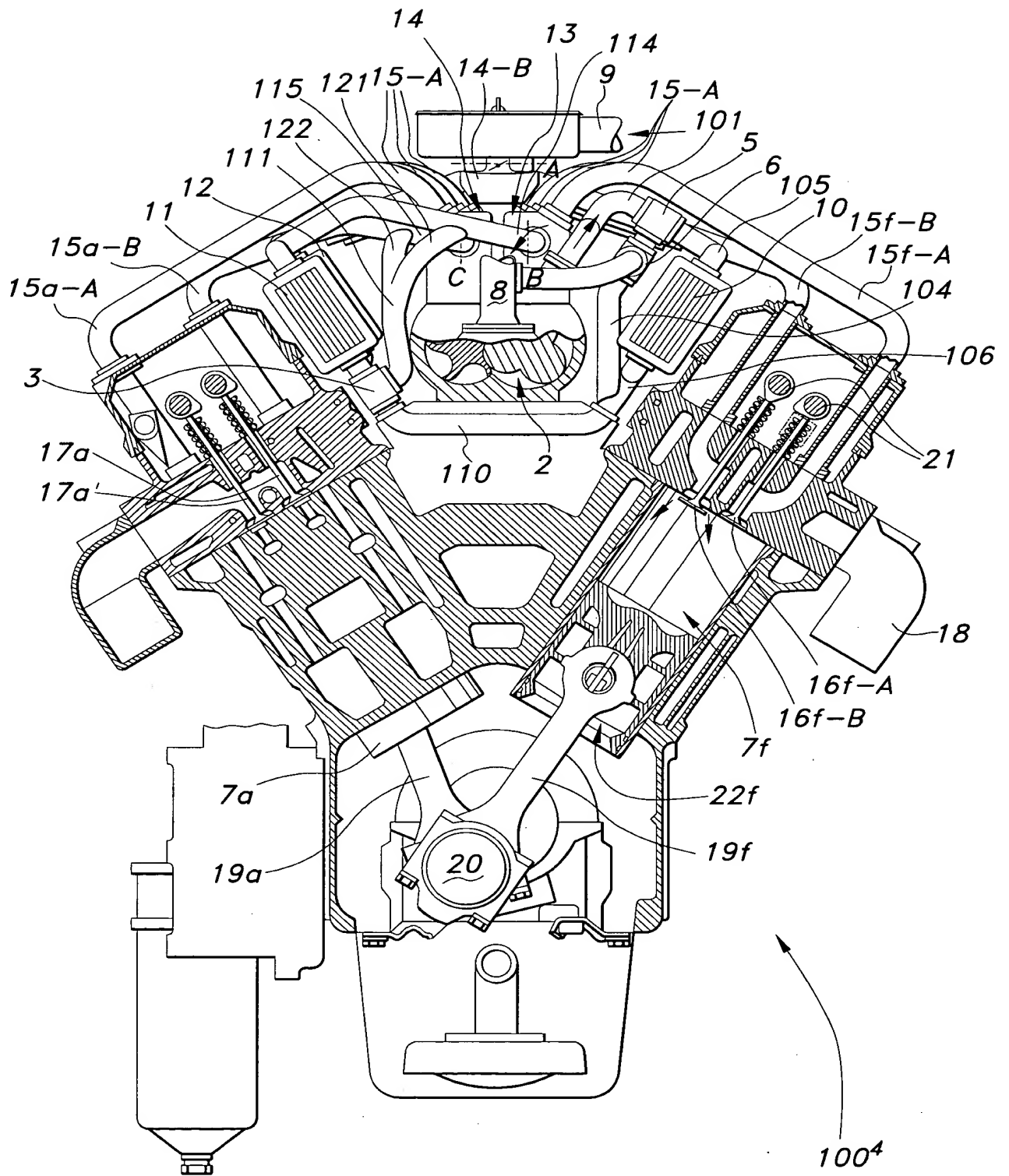


FIG 4

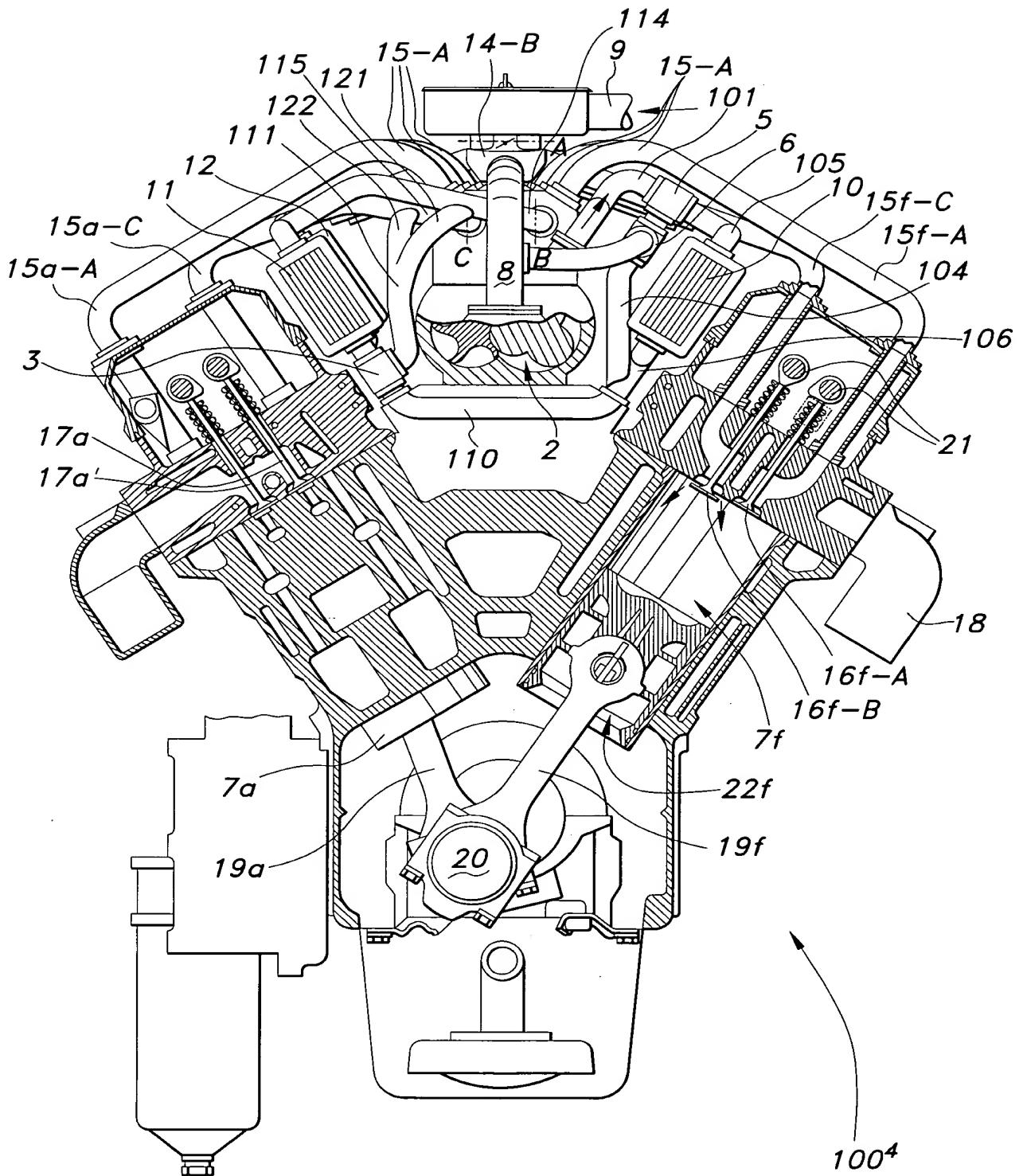


FIG 4B

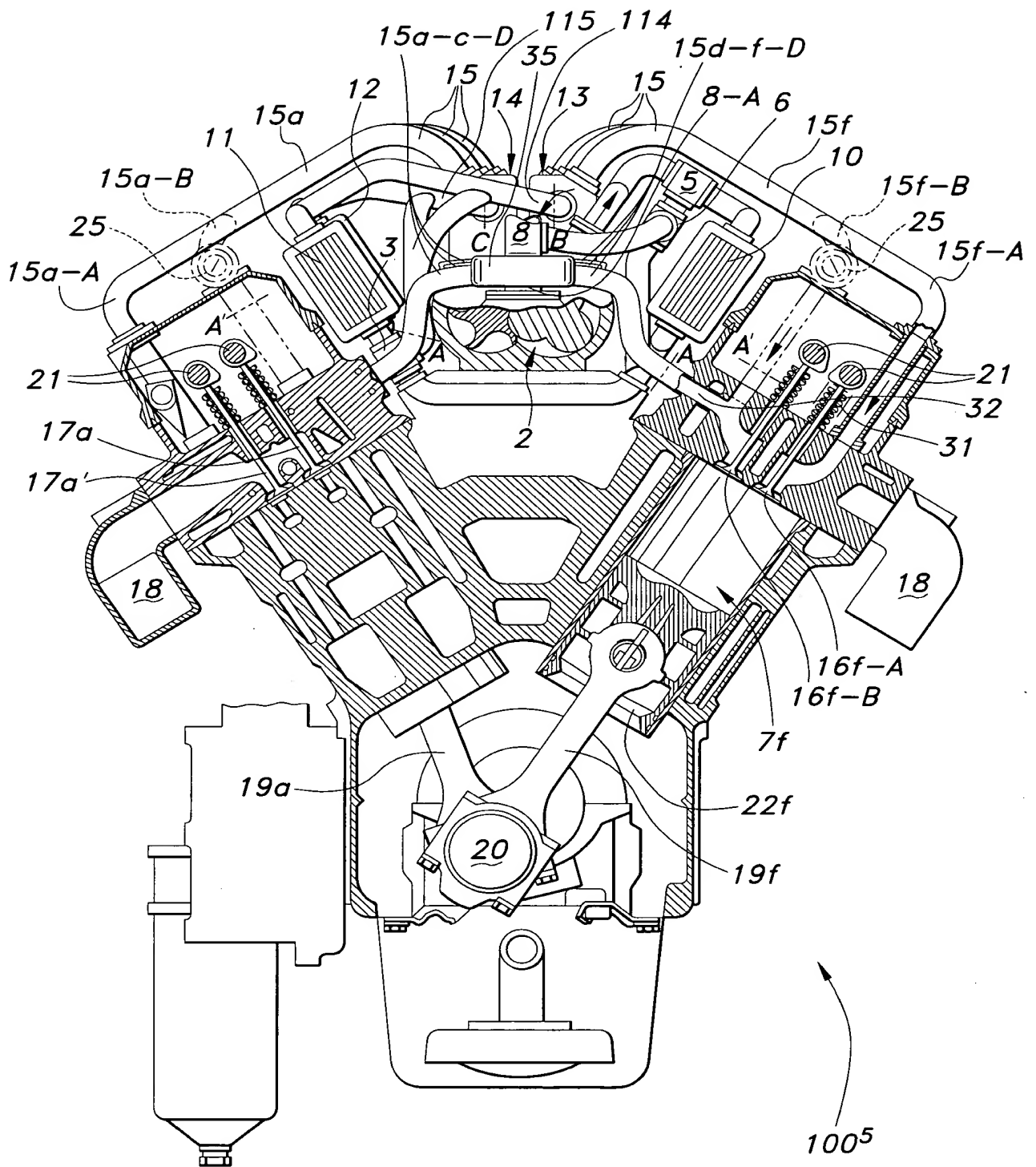


FIG 5

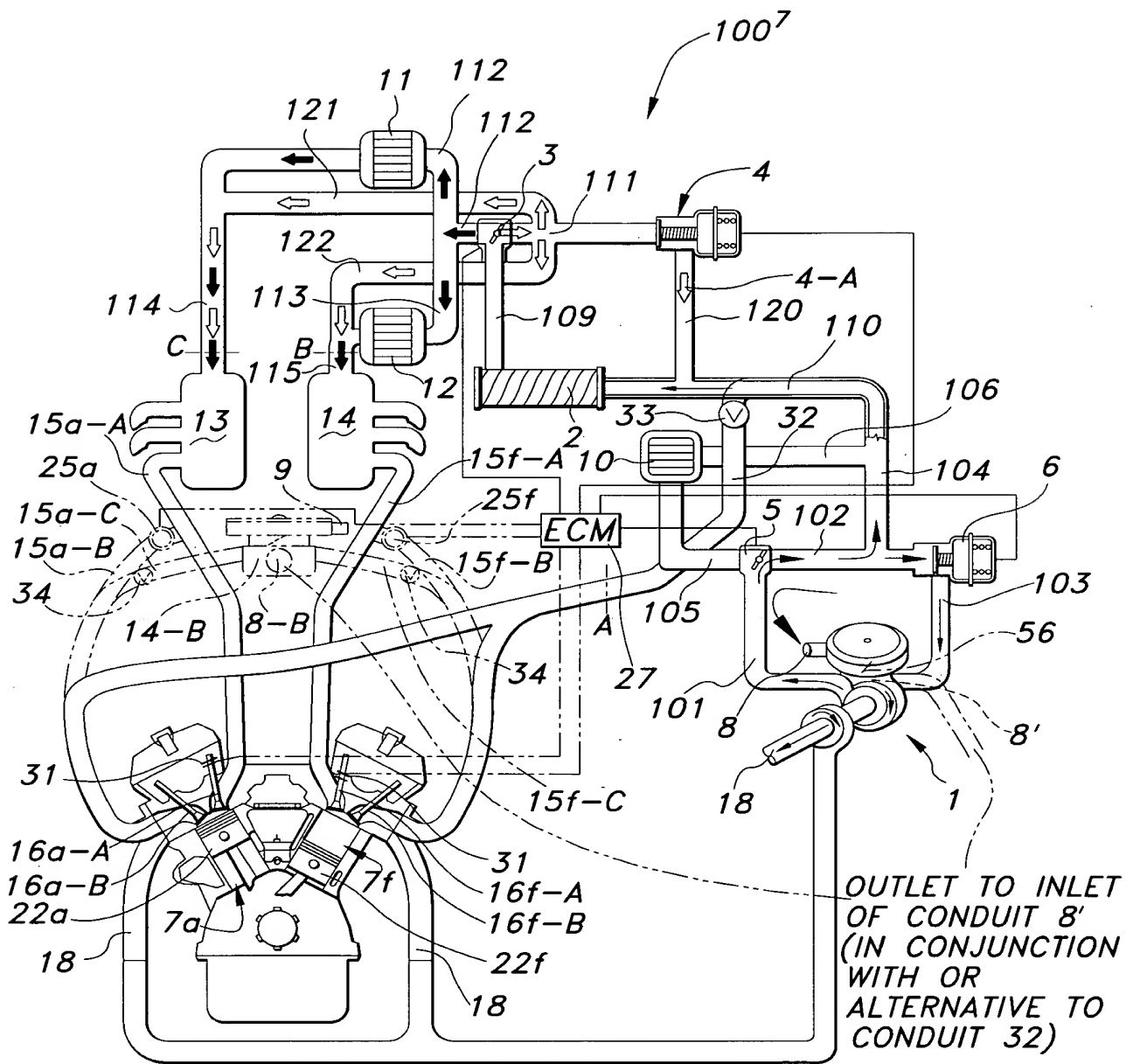


FIG 7

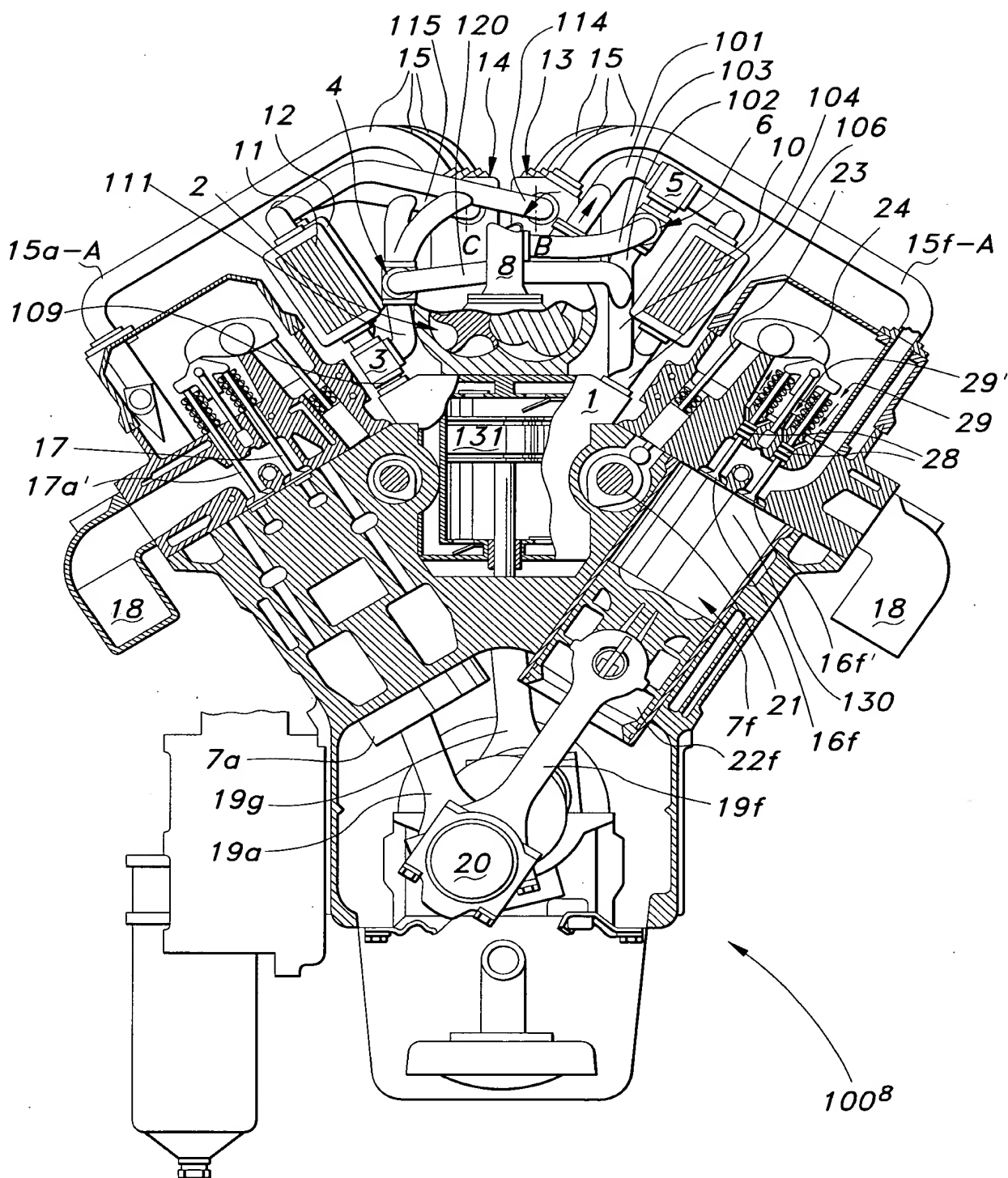


FIG 8

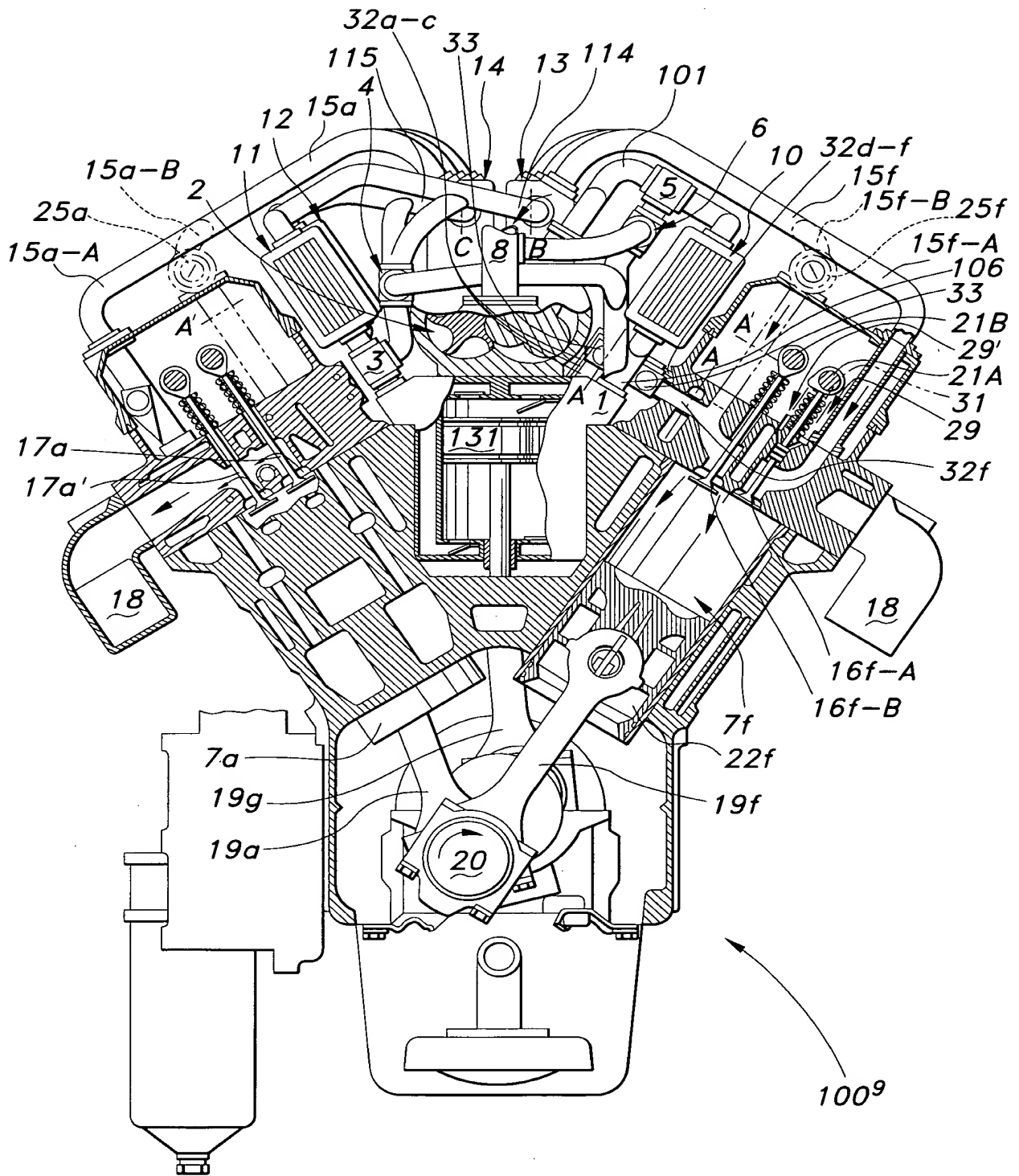


FIG 9

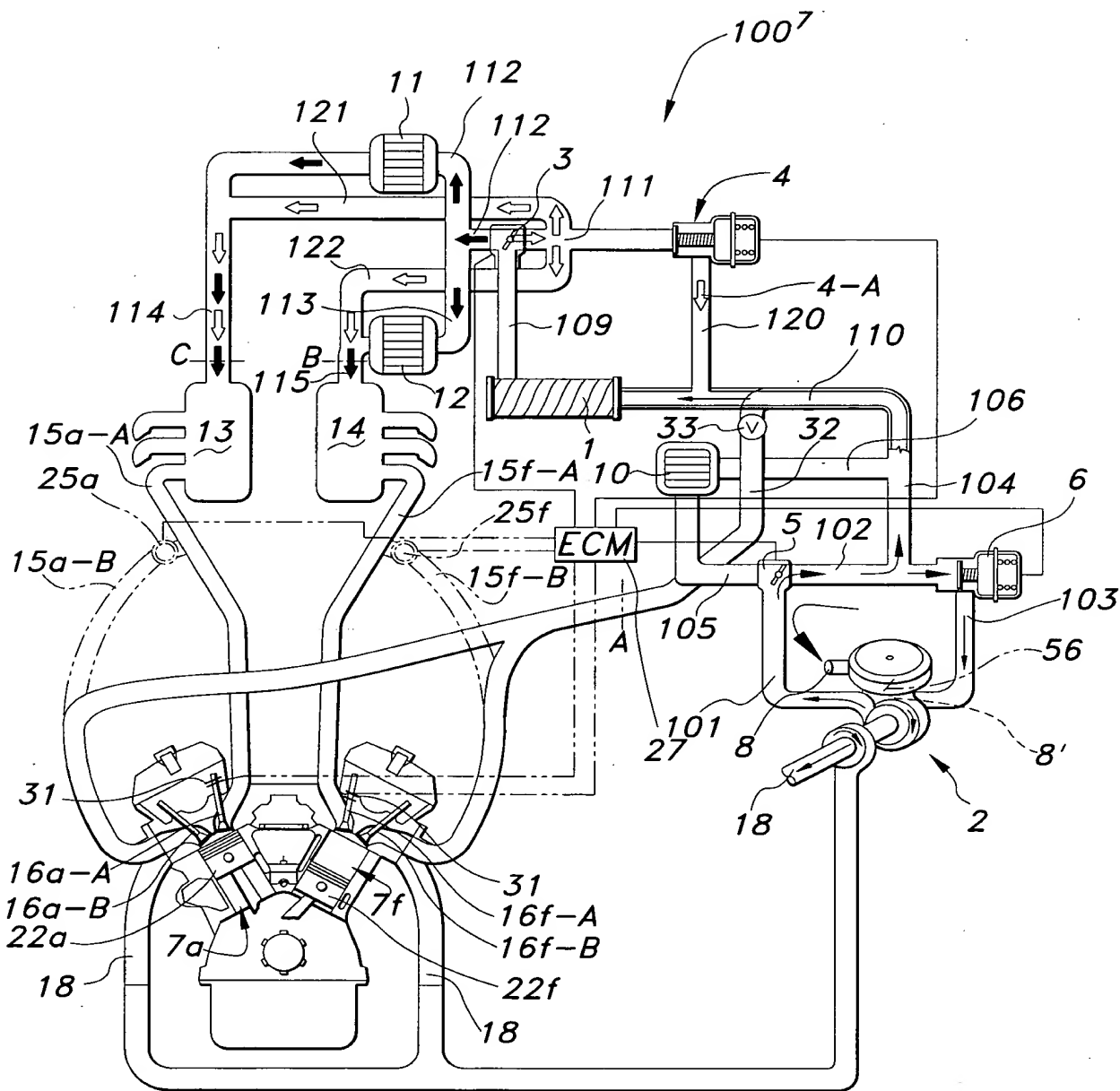


FIG 9B

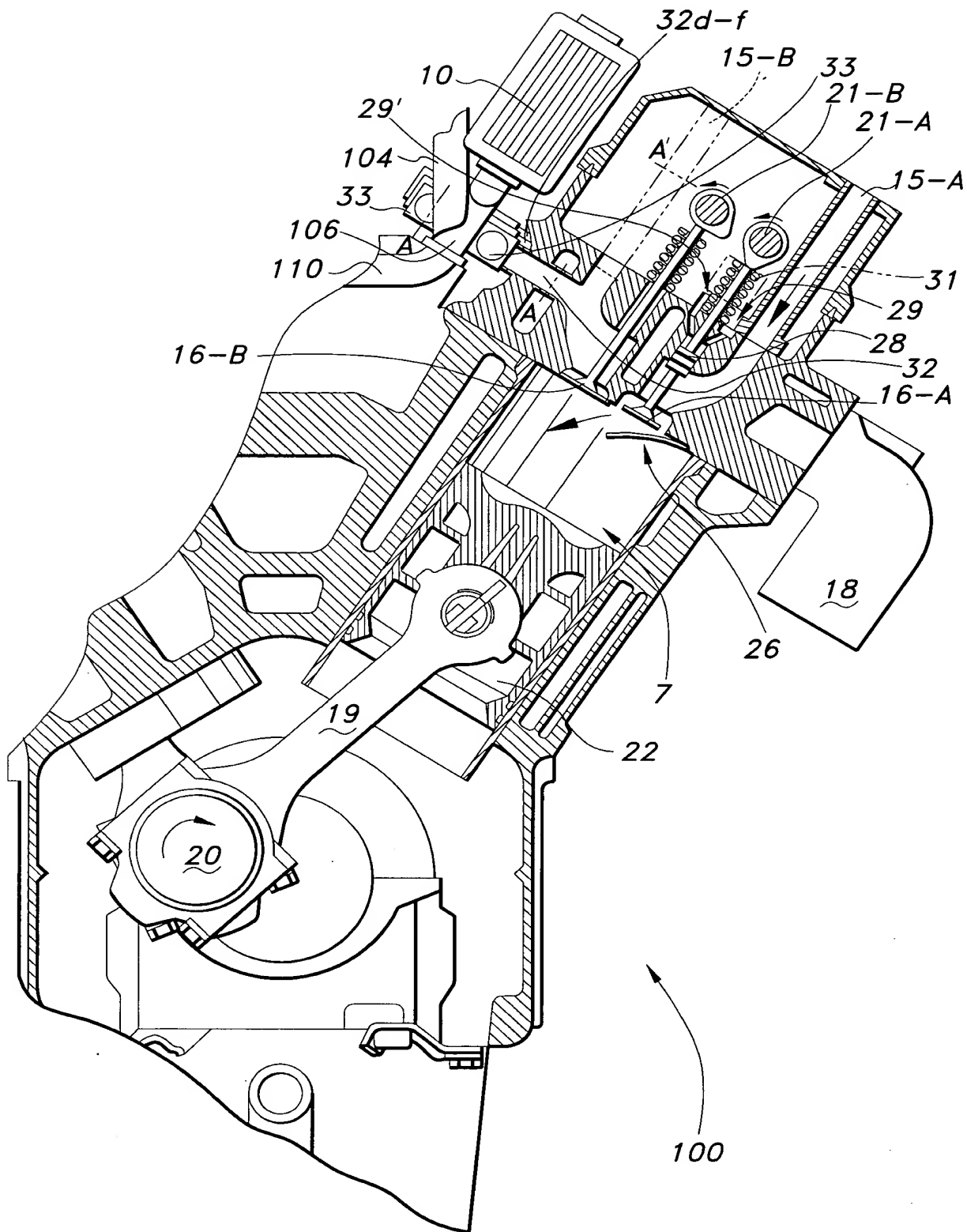


FIG 10

08863103.052397

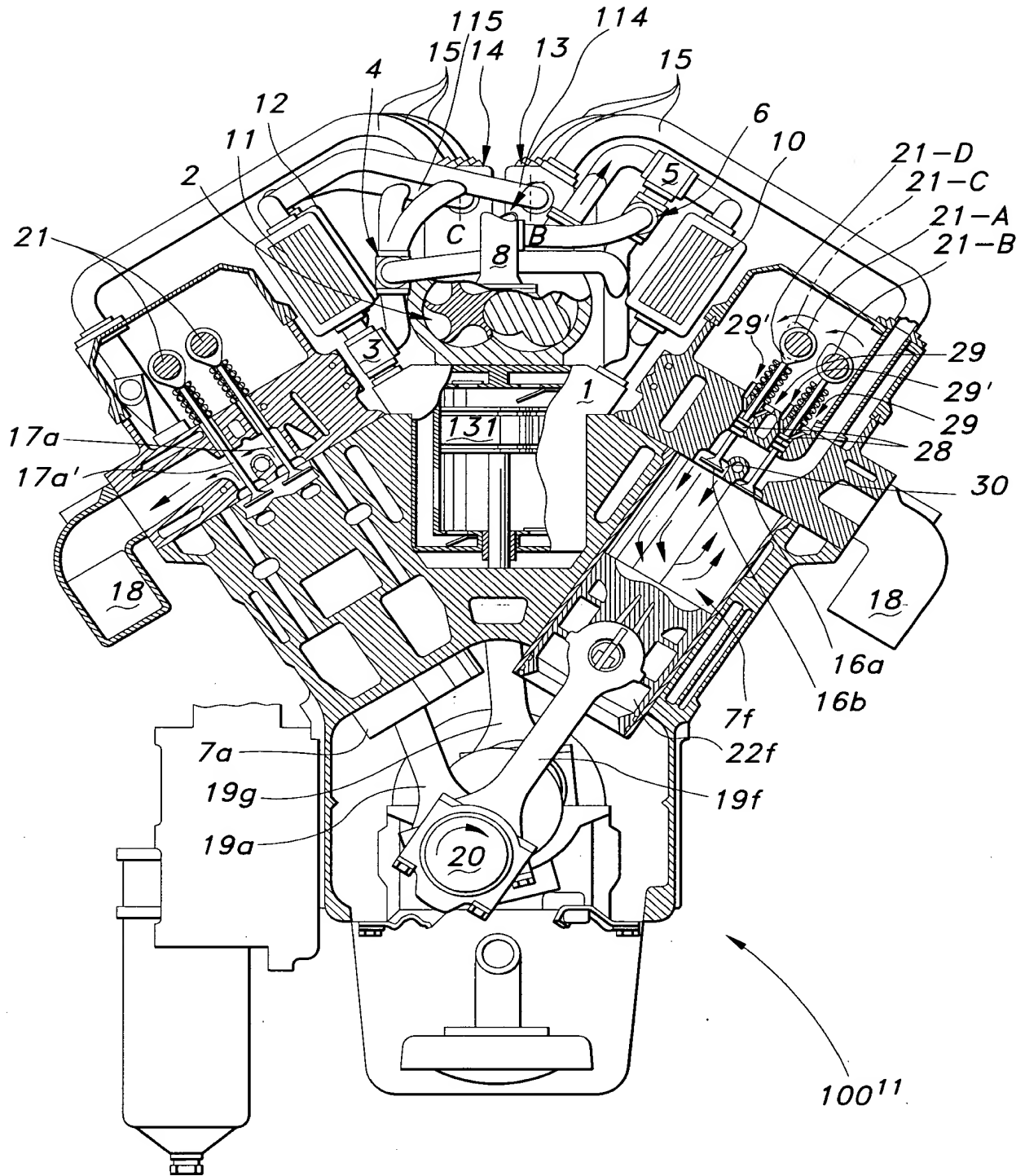
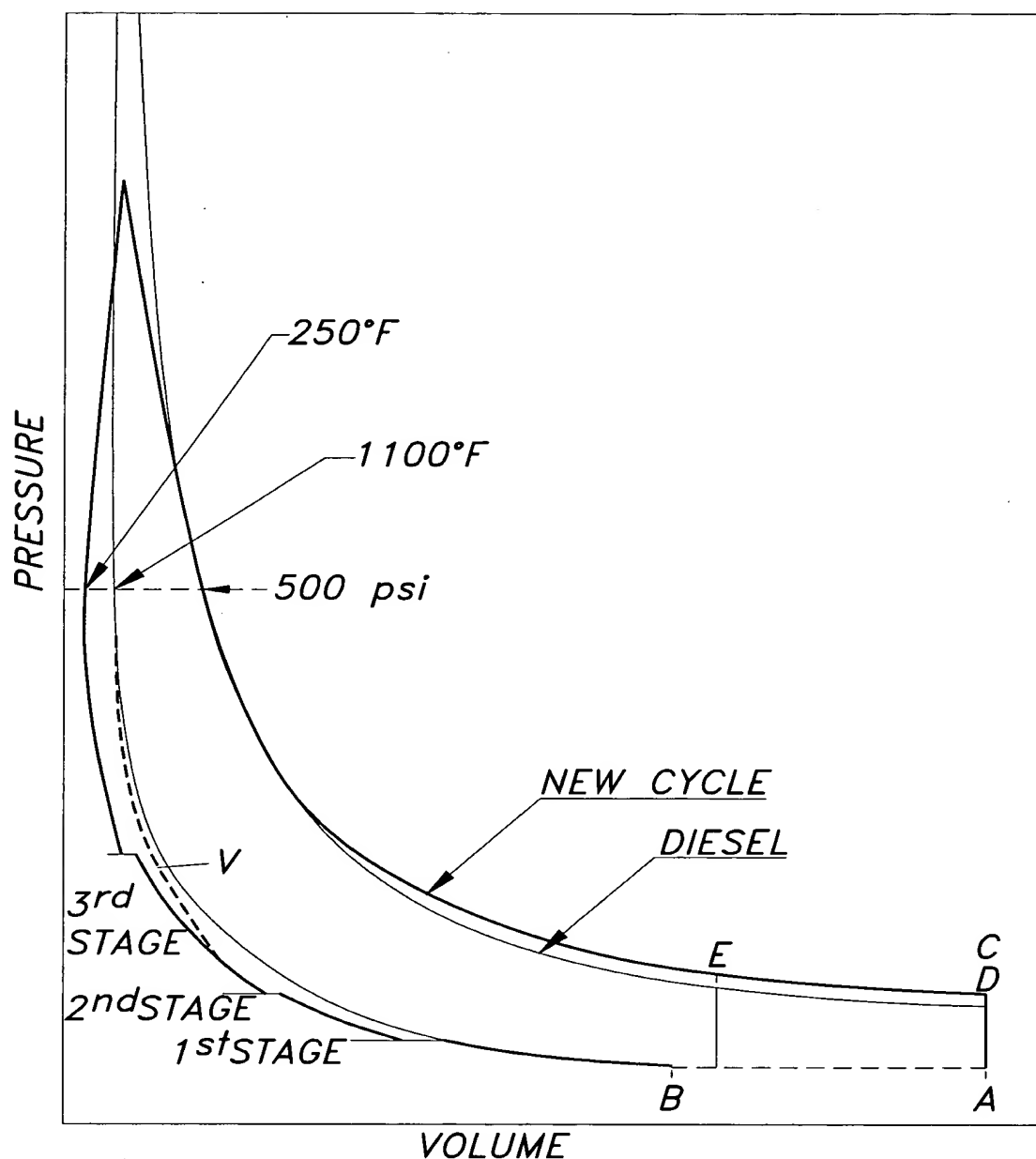
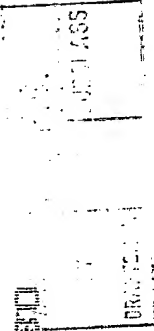


FIG 11



- A - COMPRESSION BEGINS IN 2-STROKE AND 4-STROKE DIESEL CYCLE ENGINE
- B - COMPRESSION BEGINS IN 2-STROKE AND 4-STROKE NEW CYCLE ENGINE
- C - EXPANSION ENDS IN 4-STROKE DIESEL CYCLE ENGINE
- D - EXPANSION ENDS IN 2-STROKE AND 4-STROKE NEW CYCLE ENGINE
- E - EXPANSION ENDS (AT EXHAUST BLOW-DOWN) IN 2-STROKE DIESEL CYCLE ENGINE
- V - SEE NOTE 1 IN DESCRIPTION

FIG 12



COMPARISON OF OPERATING PARAMETERS OF A HEAVY DUTY TWO-STROKE DIESEL ENGINE (A)

WITH THE ENGINE OF THIS INVENTION (B)

ENGINE	COMPRESSION RATIO OR NOMINAL COMPRESSION RATIO	EFFECTIVE COMP RATIO	COMPRESSION PRESSURE (PSI)	TEMP @ END COMP (DEG F.)	TEMP @ END COMB (DEG F.)	CHARGE DENSITY (LB./CU. FT.)	EXPANSION RATIO	E. R. C. R.	CHARGE WEIGHT PER REVOLUTION (GRAMS)
A	19:1	19:1	907	1300	3400	1.45	*10:1	0.5	2.06
B(ic)	13:1	2:1	533	250	3000	2.03	**19:1	1.5	2.86
B(bp)	13:1	13:1	533	992	^3100	1.01	**19:1	1.5	1.43
B2(ic)	10:1	2:1	369	250	^2800	1.40	**19:1	1.9	1.98
B2(bp)	10:1	10:1	369	871	^2900	0.75	**19:1	1.9	1.06

* Exhaust valve opens midstroke

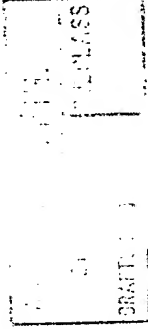
** Exhaust valve opens near BDC

(ic) Air charge intercooled except for last stage of compression

(bp) Intercoolers bypassed

^ Estimated

E. R. = EXPANSION RATIO
C. R. = COMPRESSION RATIO



COMPARISON OF OPERATING PARAMETERS OF A HEAVY DUTY FOUR-STROKE DIESEL ENGINE (A)

WITH THE ENGINE OF THIS INVENTION (B)

ENGINE	COMPRESSION RATIO OR NOMINAL COMPRESSION RATIO	EFFECTIVE COMP RATIO	COMPRESSION PRESSURE (PSI)	TEMP @ END COMP (DEG F.)	TEMP @ END COMB (DEG F.)	CHARGE DENSITY (LB./CU. FT.)	EXPANSION RATIO	E.R. C. R.	CHARGE WEIGHT PER REVOLUTION (GRAMS)
A	19:1	19:1	907	1300	3400	1.45	19:1	1.0	*1.03
B(ic)	13:1	2:1	533	250	3000	2.03	19:1	1.5	**2.86
B(bp)	13:1	13:1	533	992	^3100	1.01	19:1	1.5	**1.43
B2(ic)	10:1	2:1	369	250	^2800	1.40	19:1	1.9	**1.98
B2(bp)	10:1	10:1	369	871	^2900	0.75	19:1	1.9	**1.06

* Per revolution, not per firing stroke

** Per revolution and per firing stroke

(ic) Air charge intercooled except for last stage of compression

(bp) Intrecoolers bypassed

^ Estimated

E.R. = $\frac{\text{EXPANSION RATIO}}{\text{COMPRESSION RATIO}}$

FIG 14

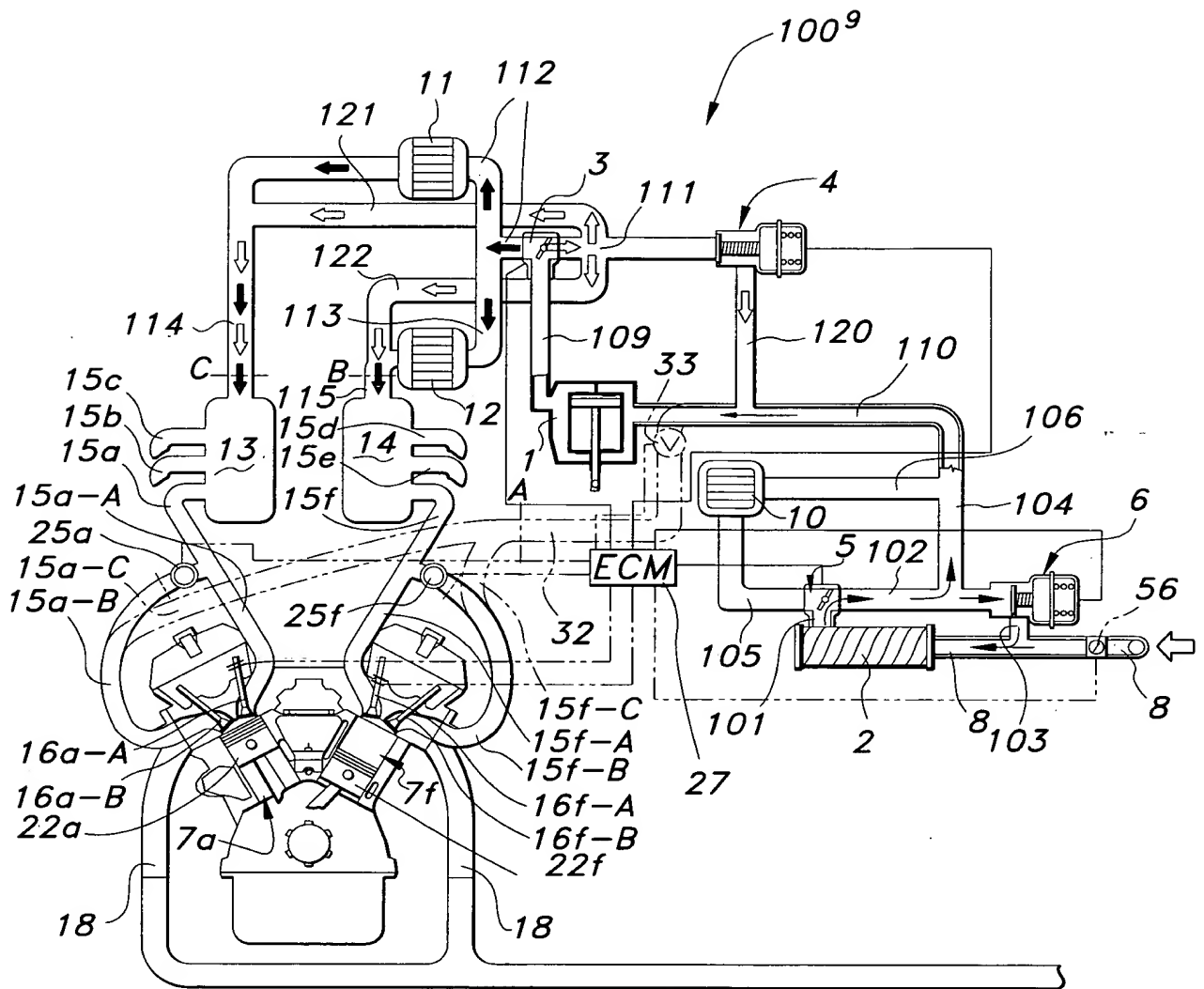


FIG 15

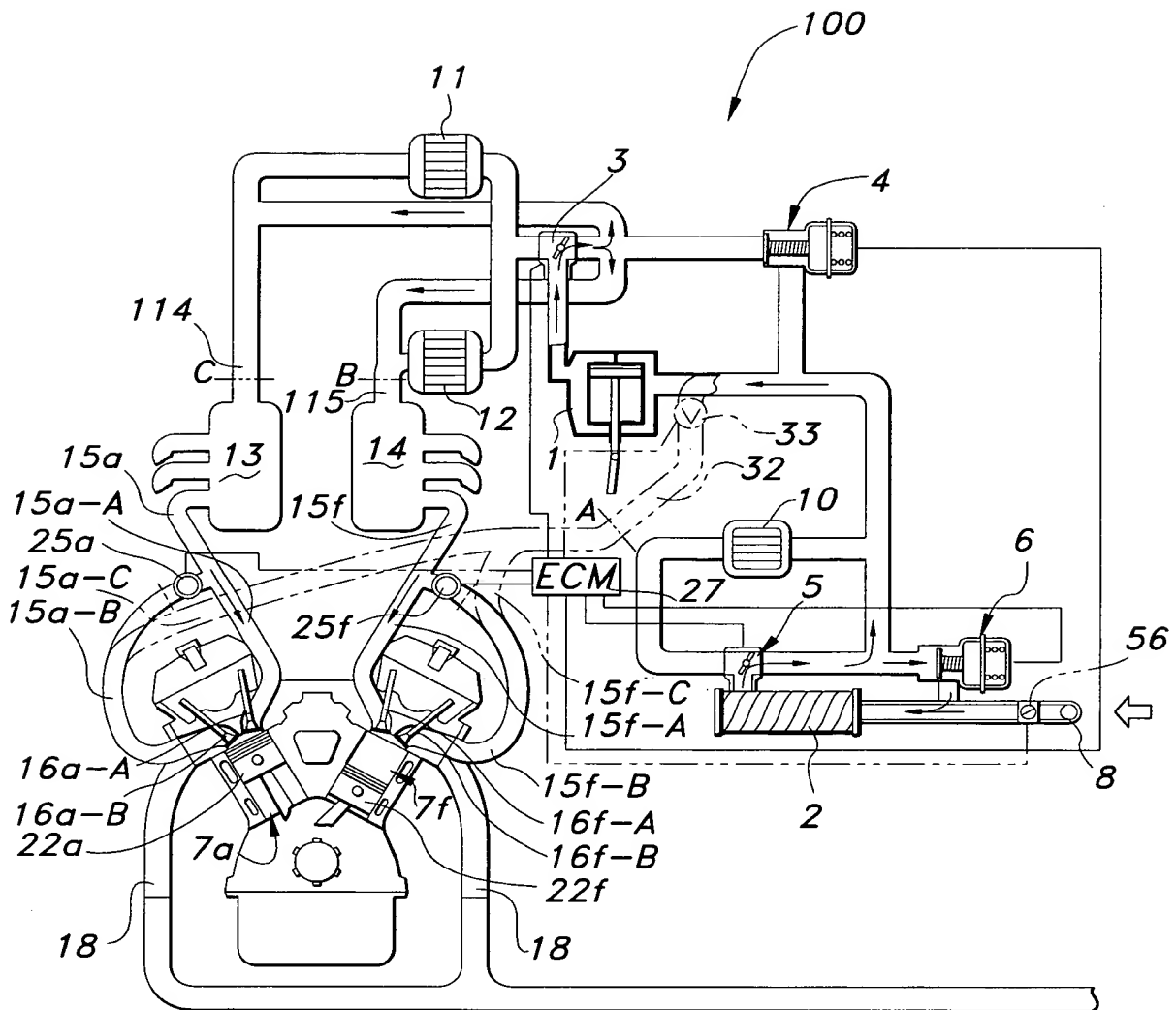


FIG 16

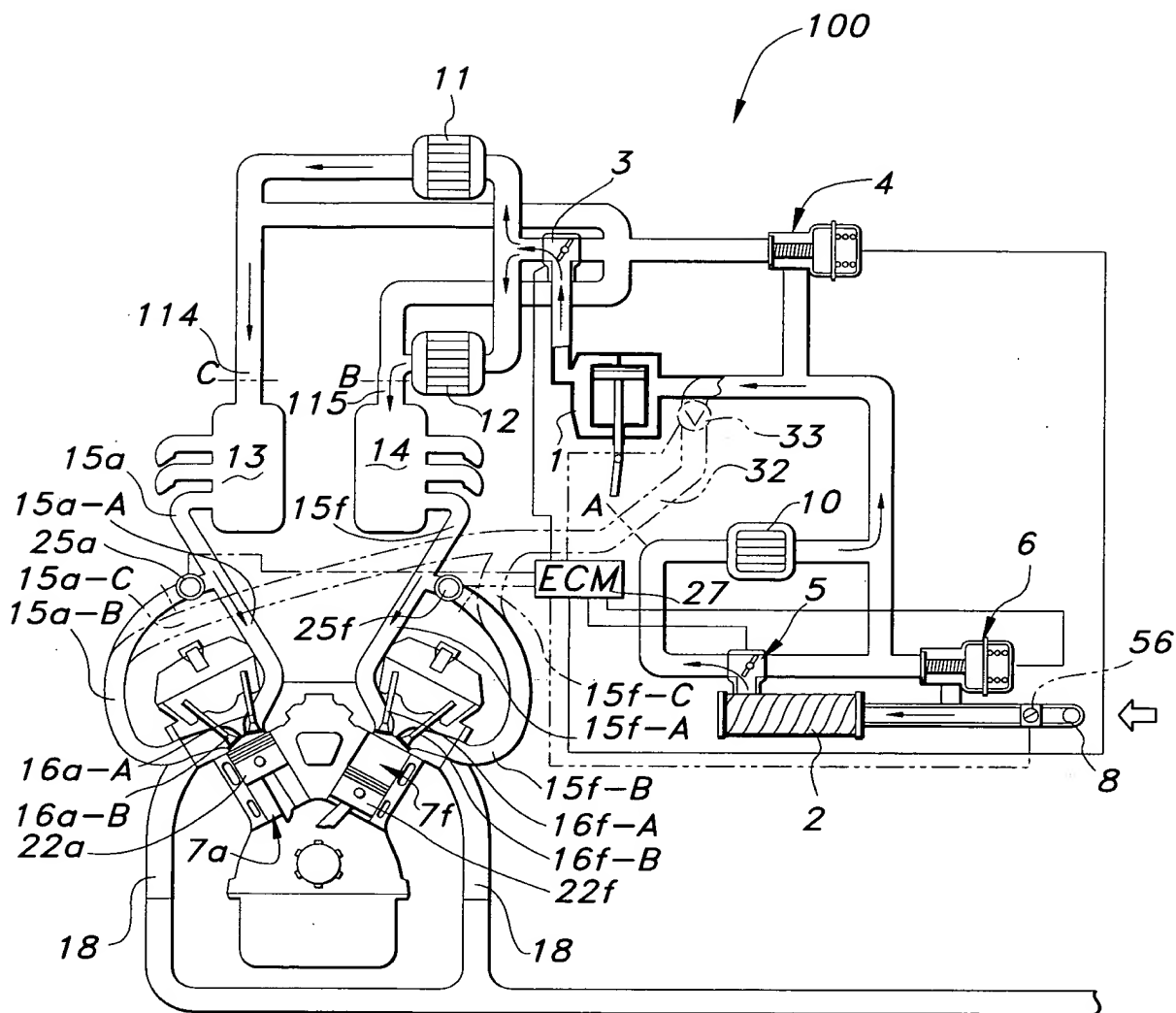


FIG 17

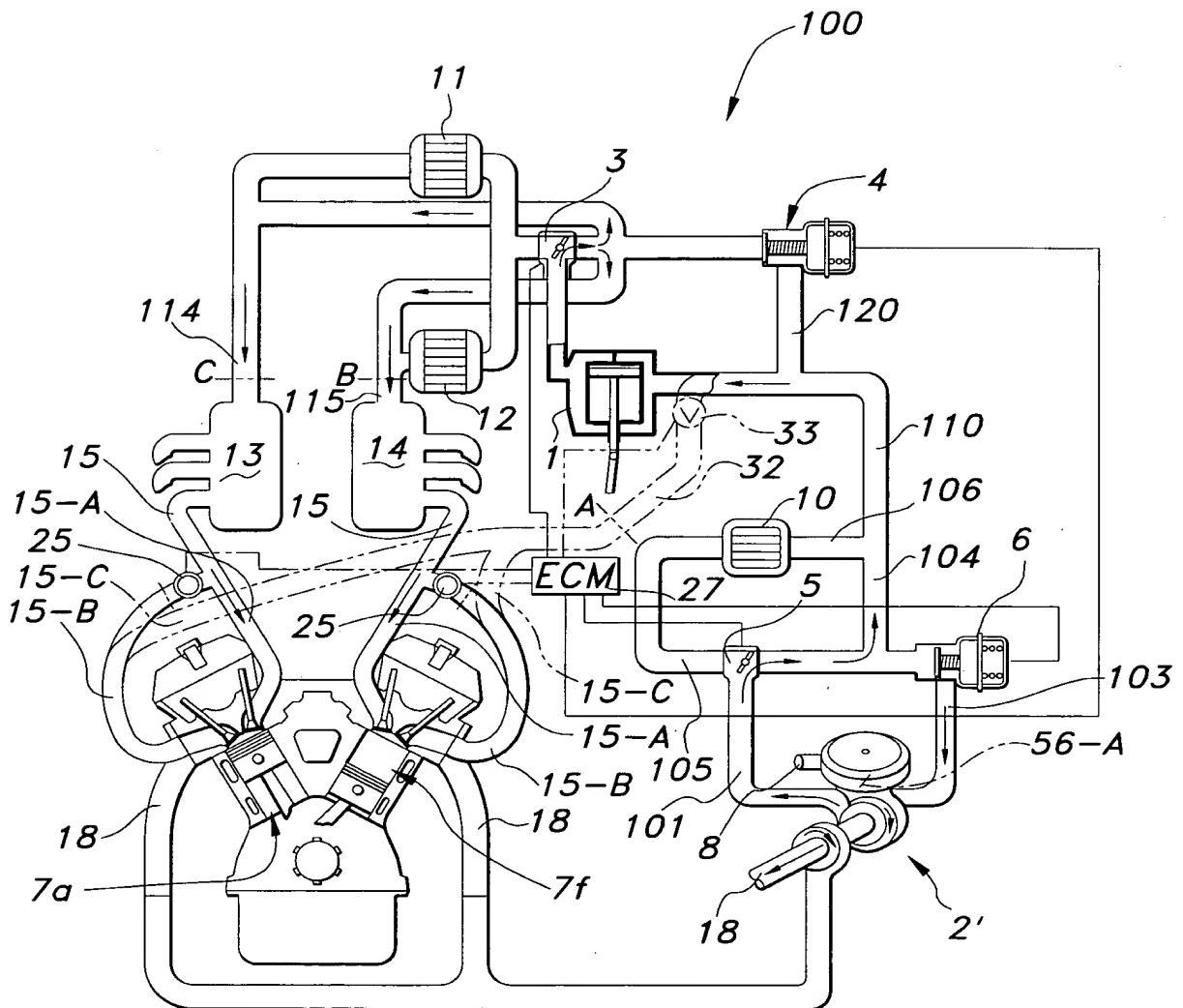


FIG 18

08663103-05299
 DRAFT

08663103-05299

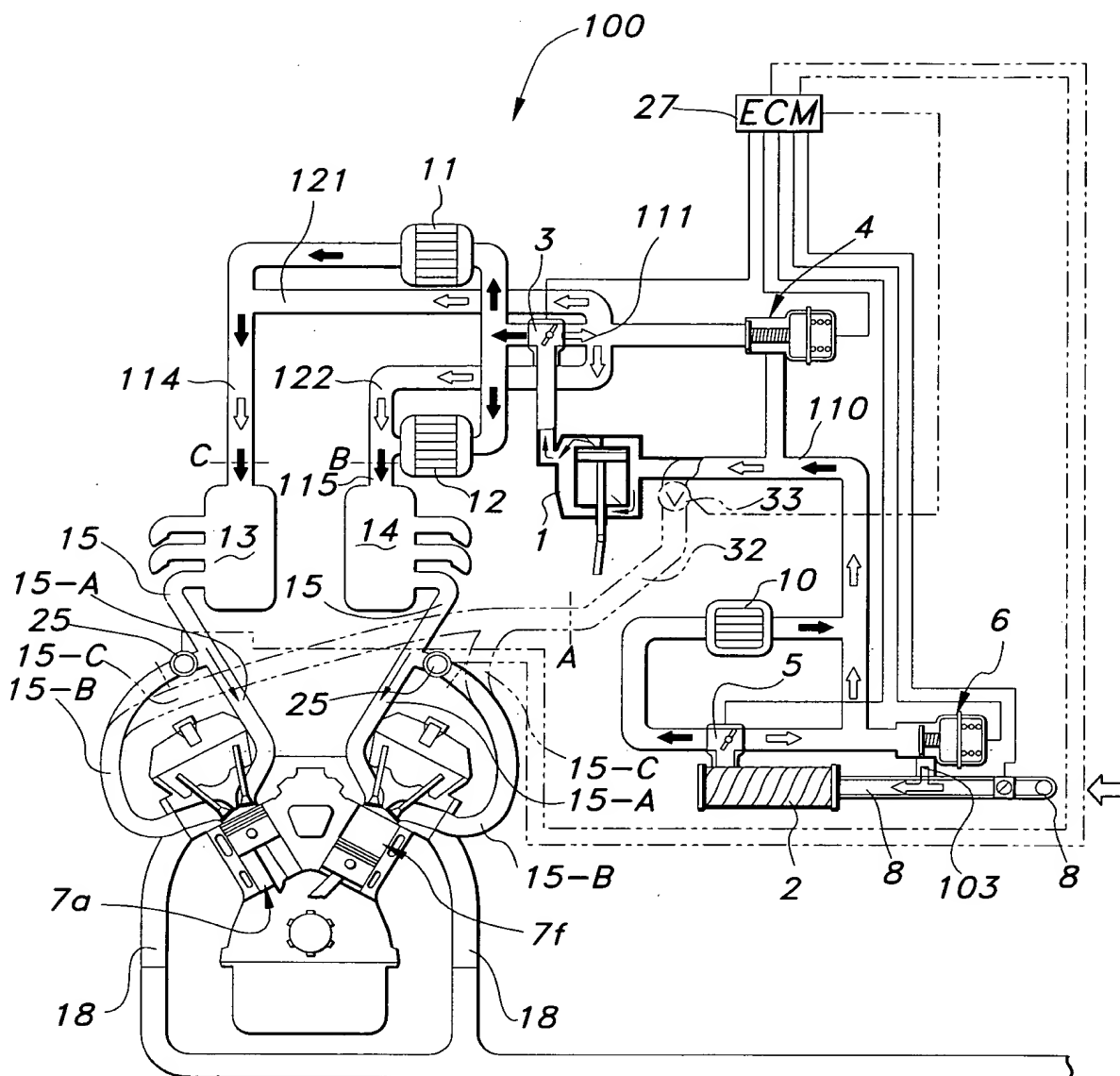
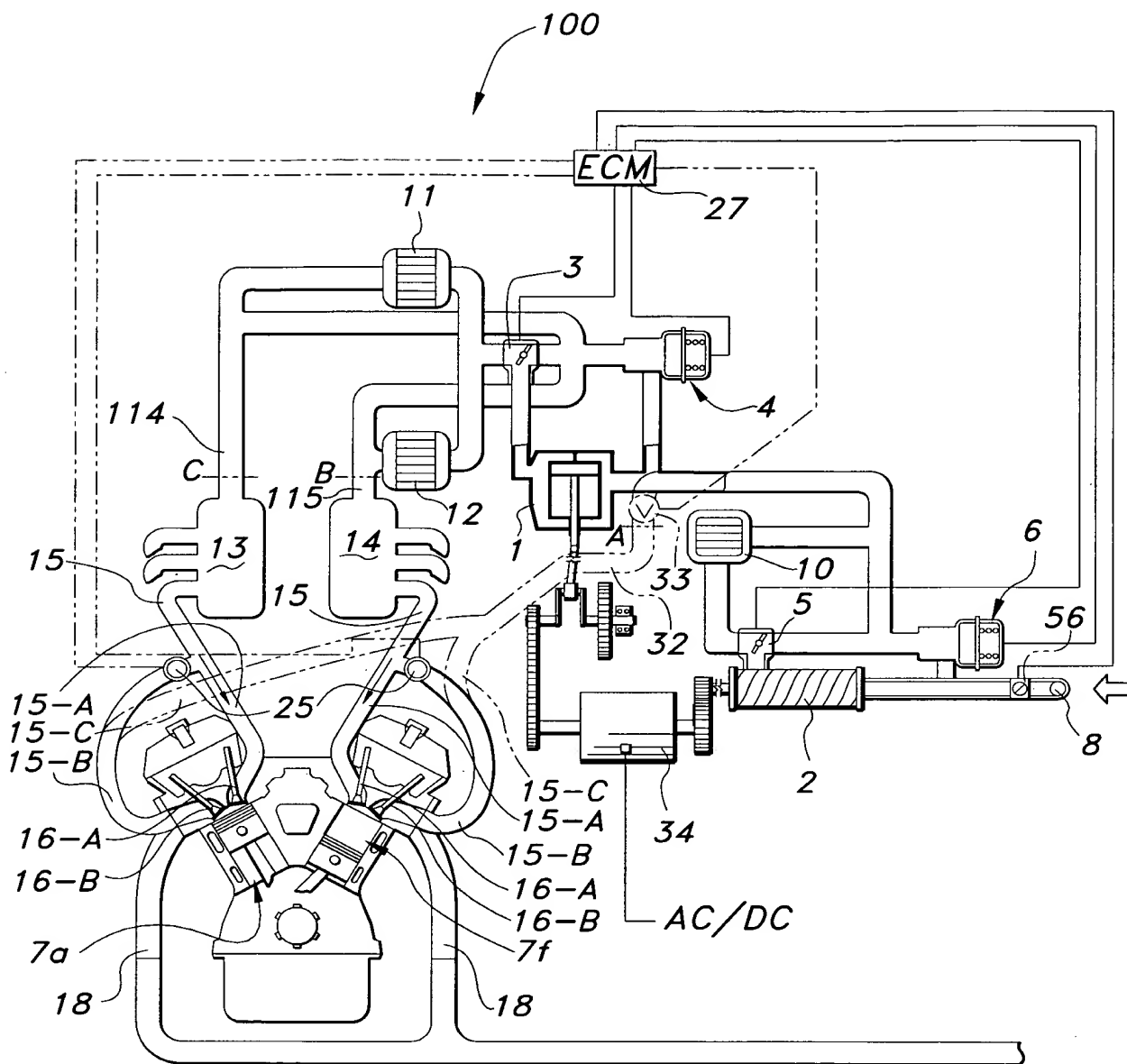


FIG 19



08663103-0523493
 08663103-0523493
 08663103-0523493

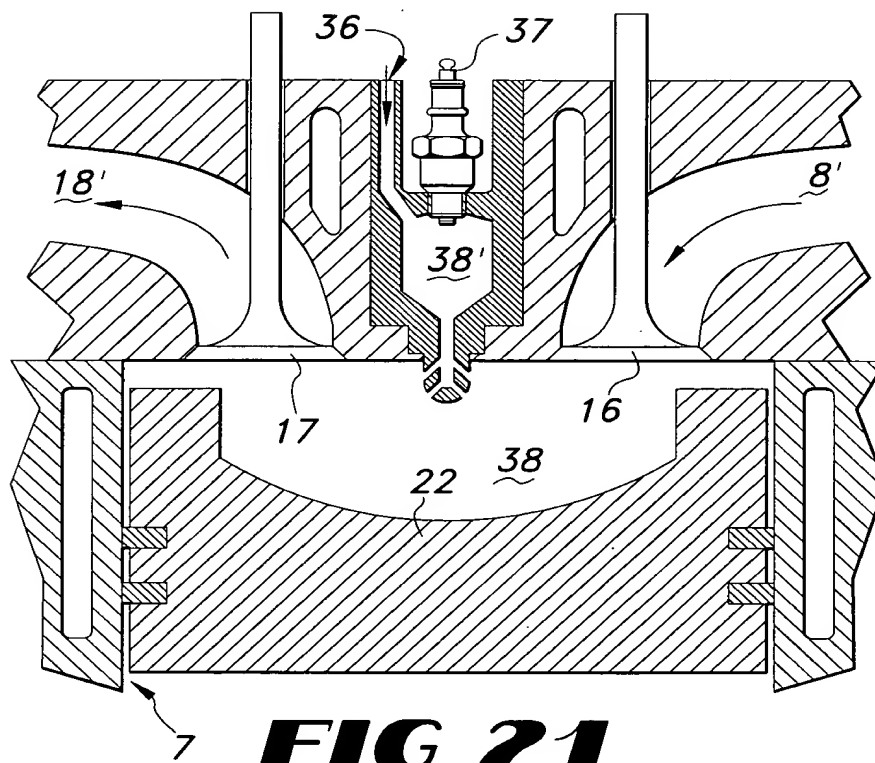


FIG 21

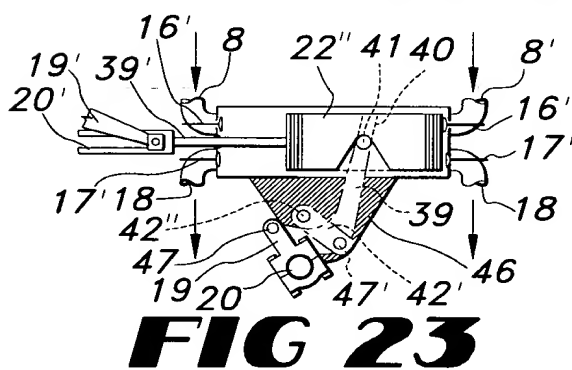


FIG 23

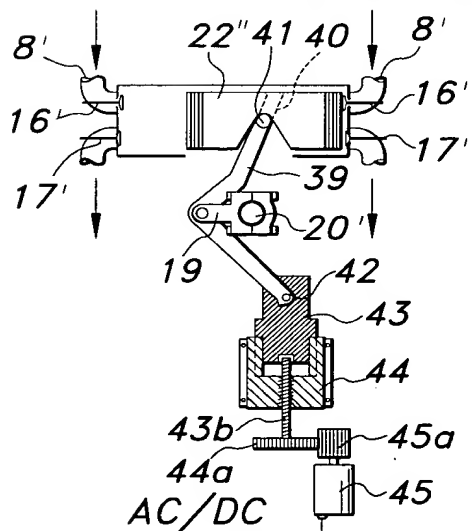


FIG 22

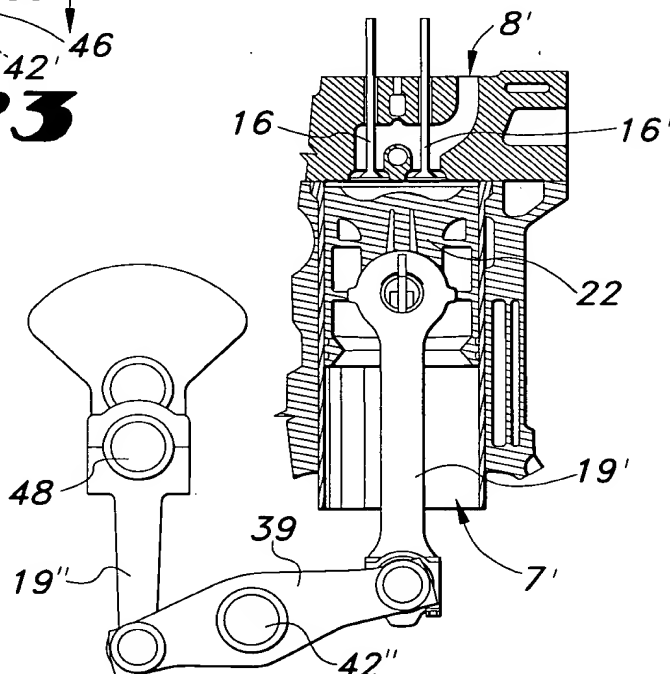


FIG 24

45250-009880
CLASS
DRAWING

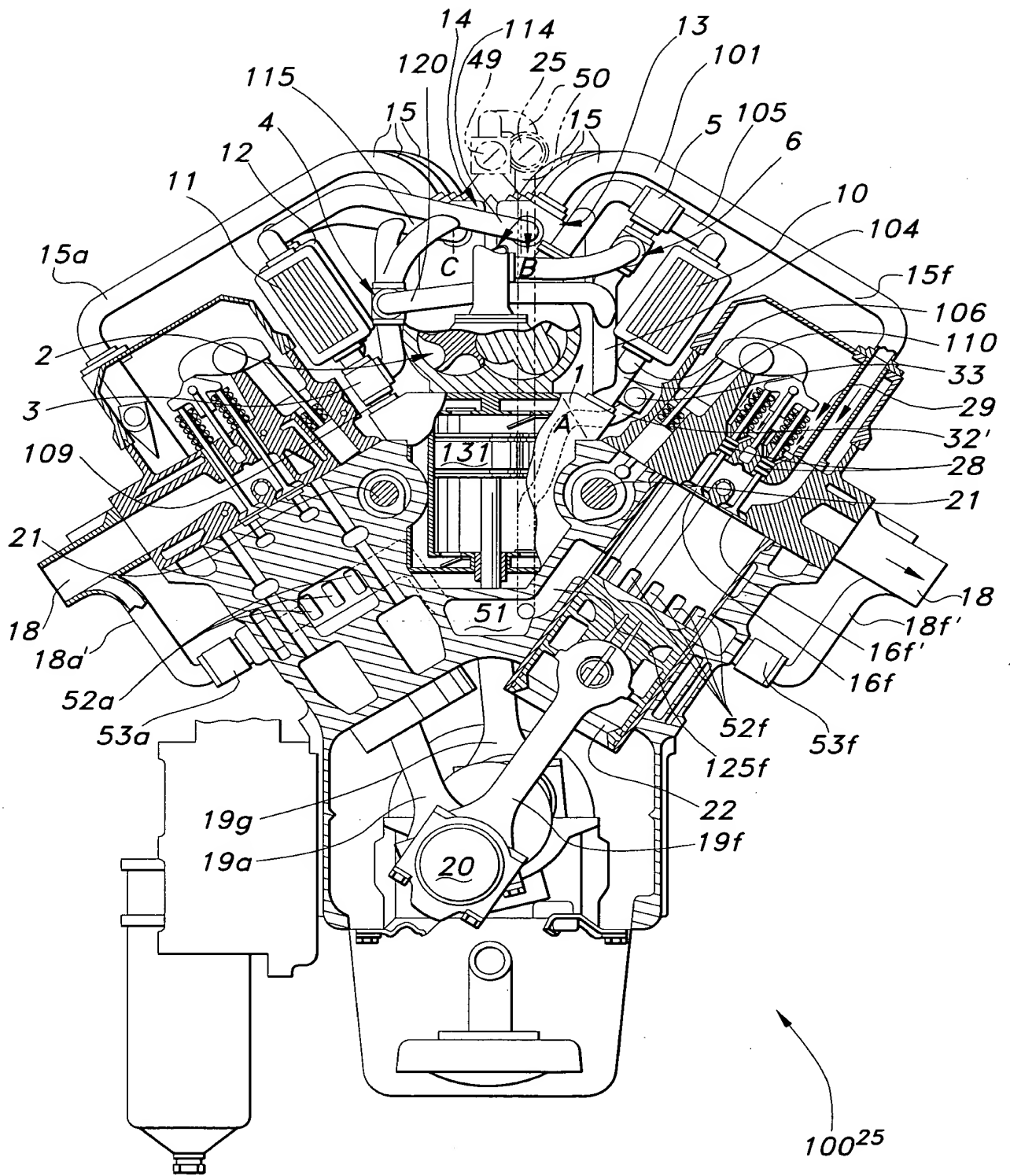


FIG 25

46220 "EOTF930"
CLASS
DRAFT

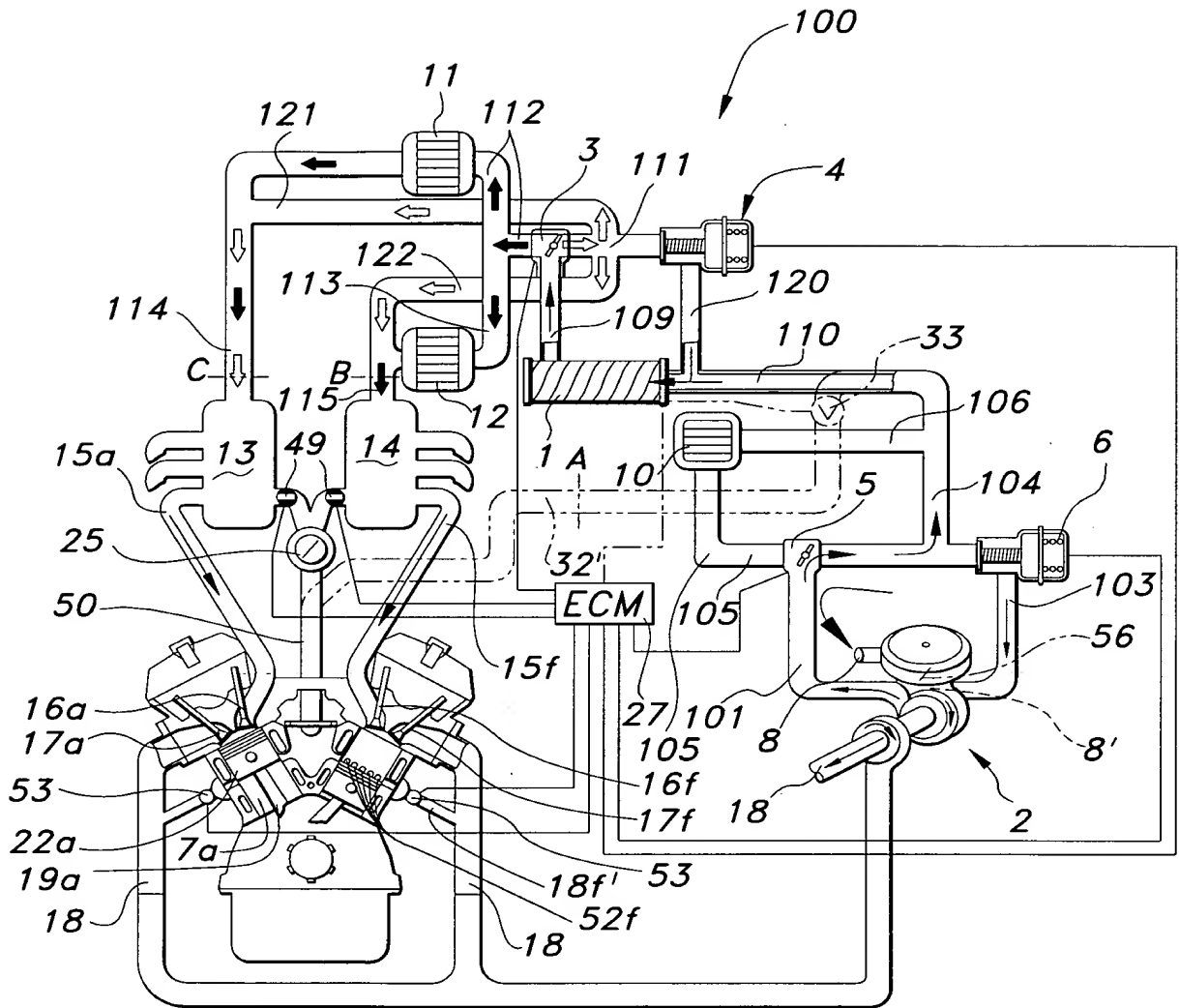


FIG 26

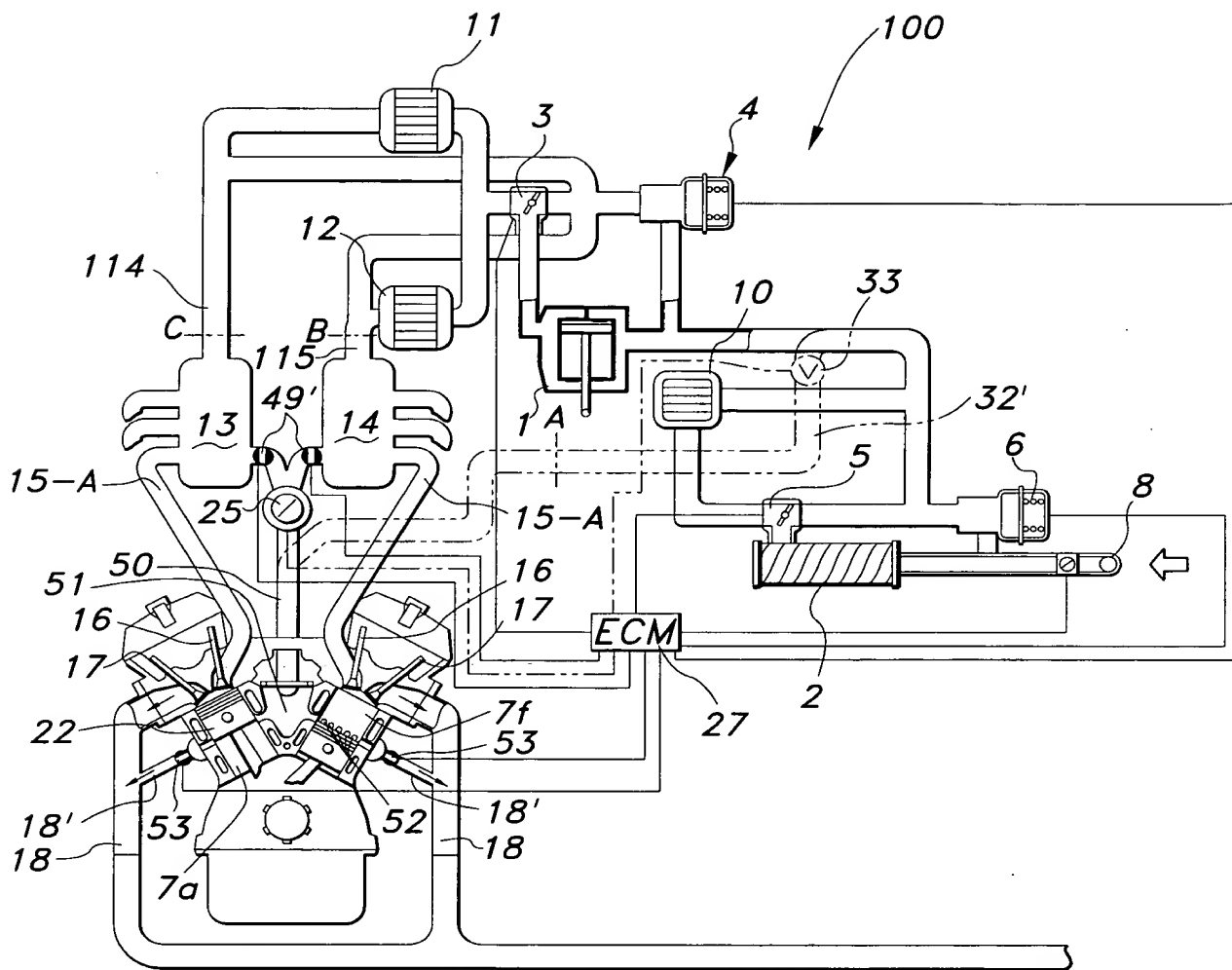


FIG 27

FIG 28

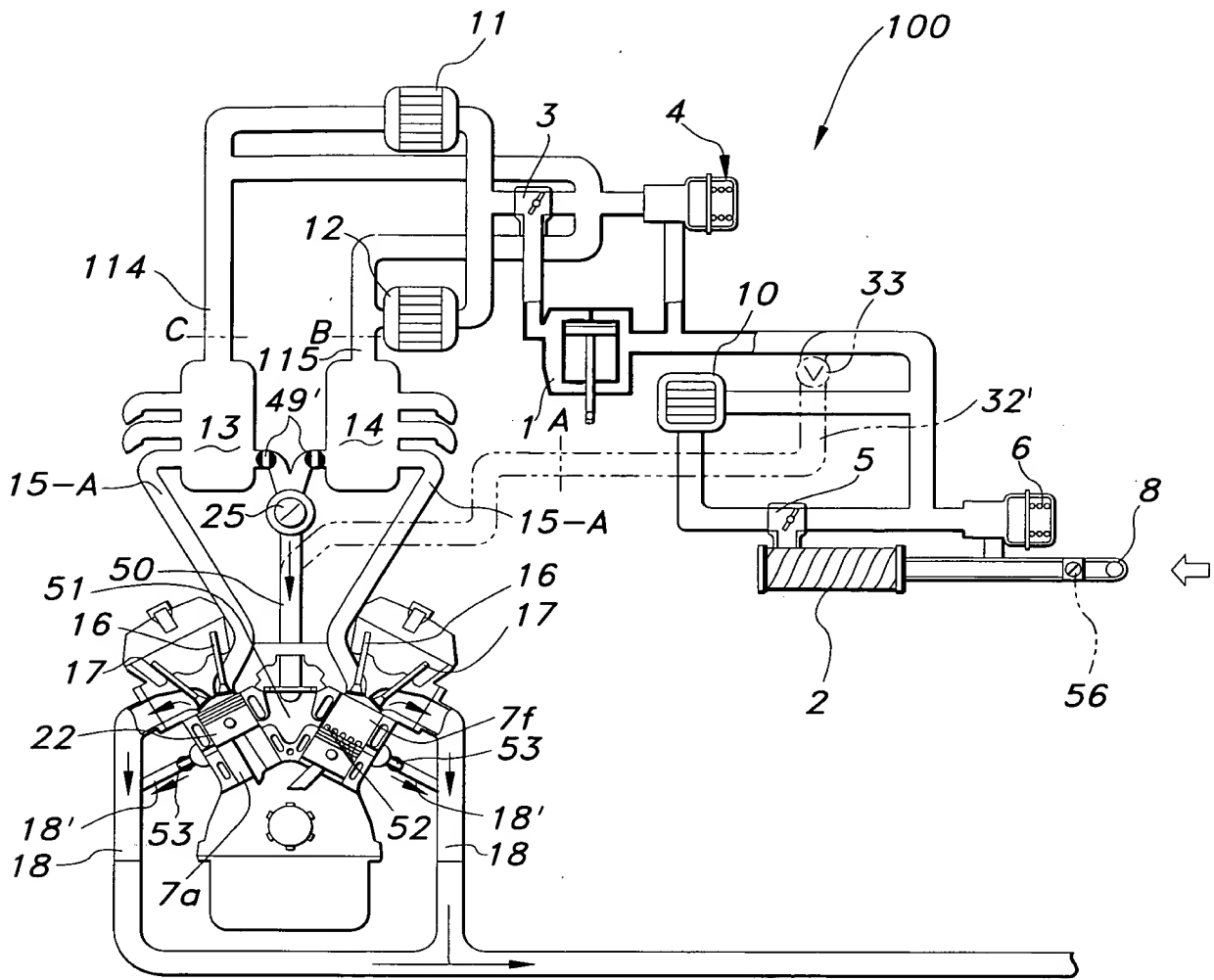


FIG 29

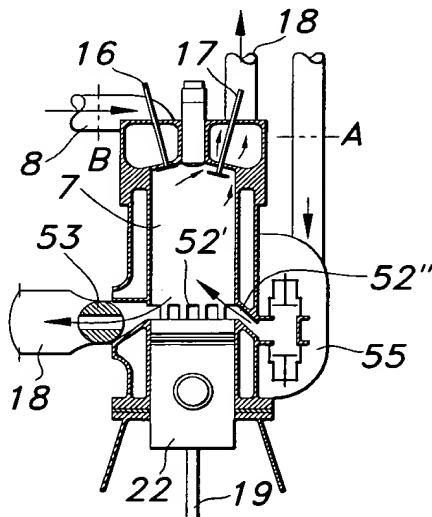


FIG 30

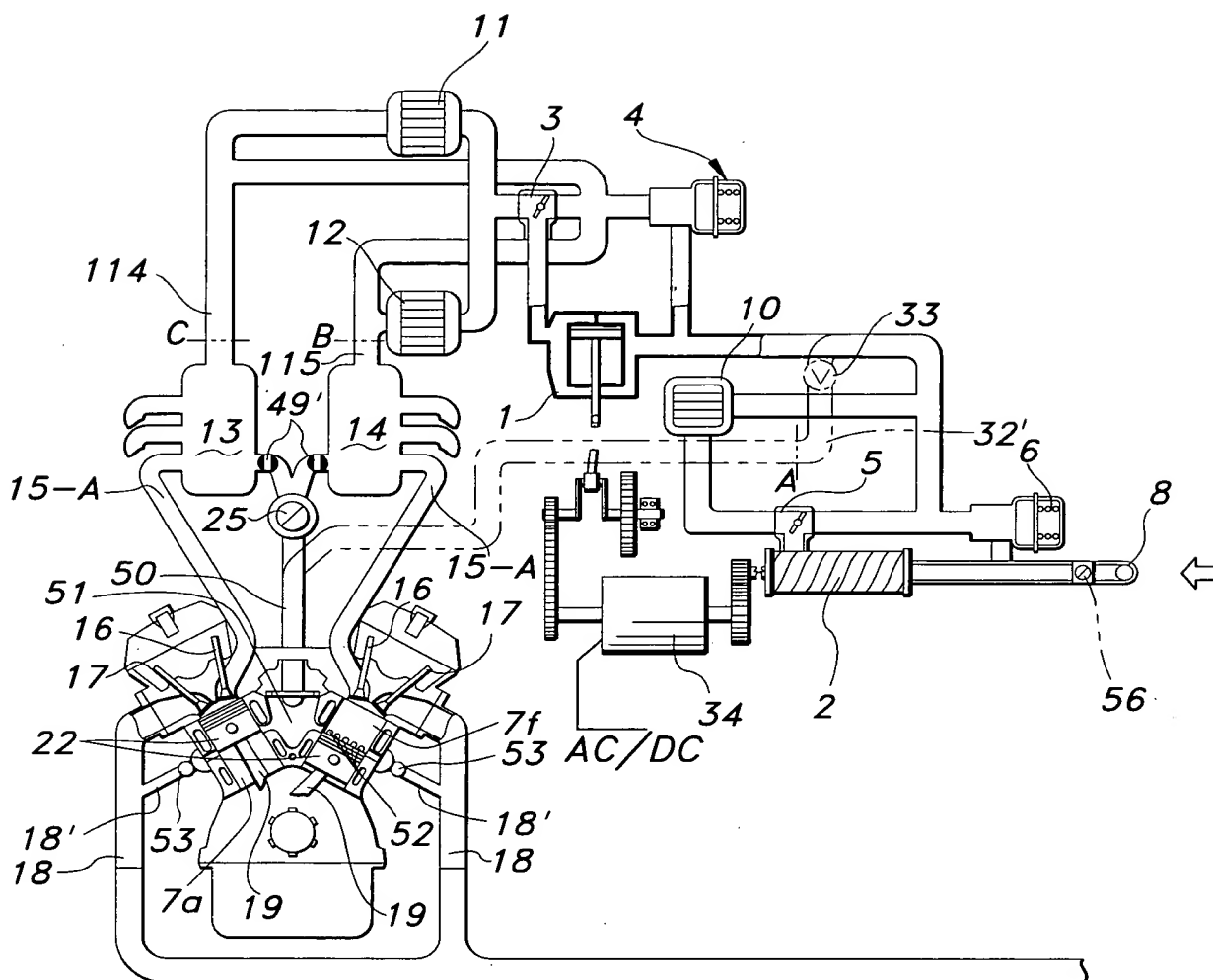


FIG 31

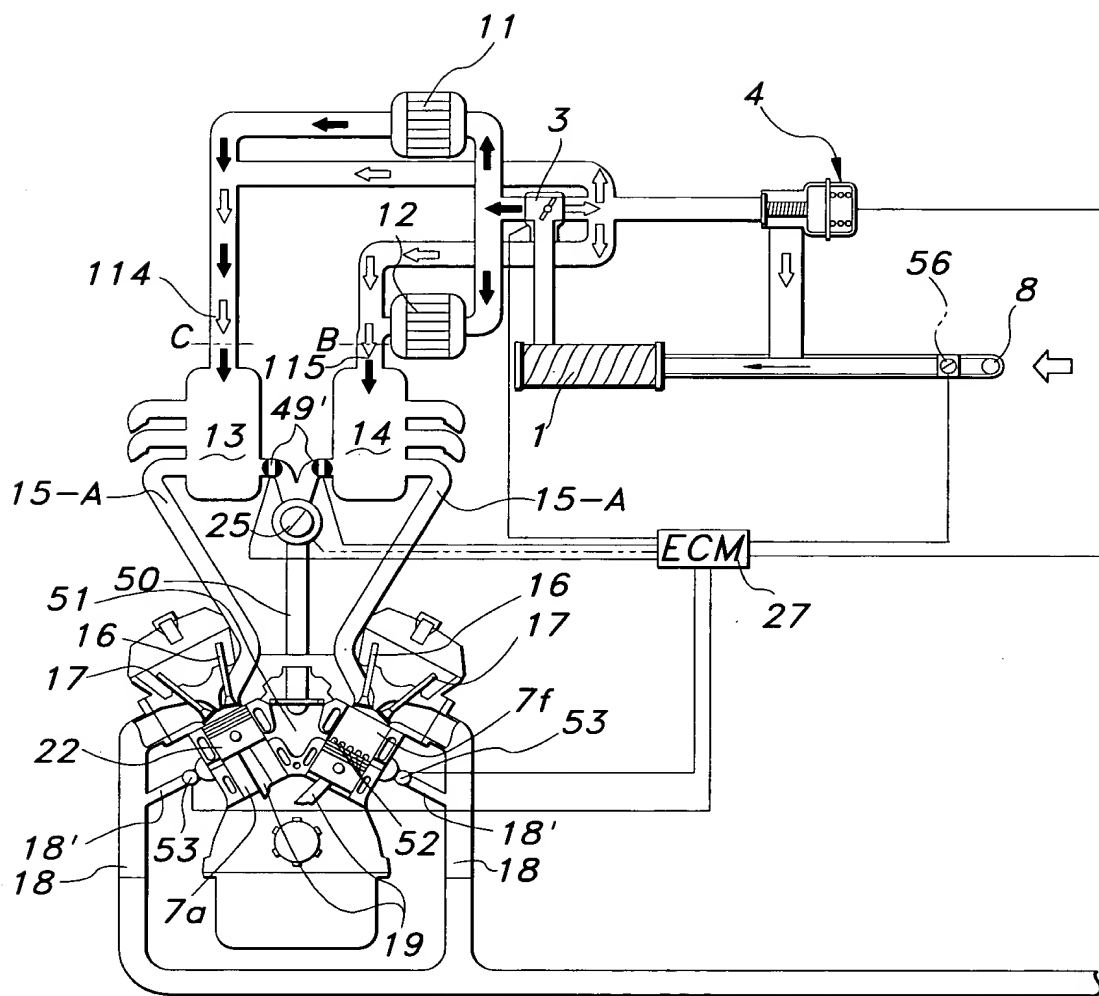


FIG 32

466250" EOT E9880

UNCLASS
DRAFTS
17

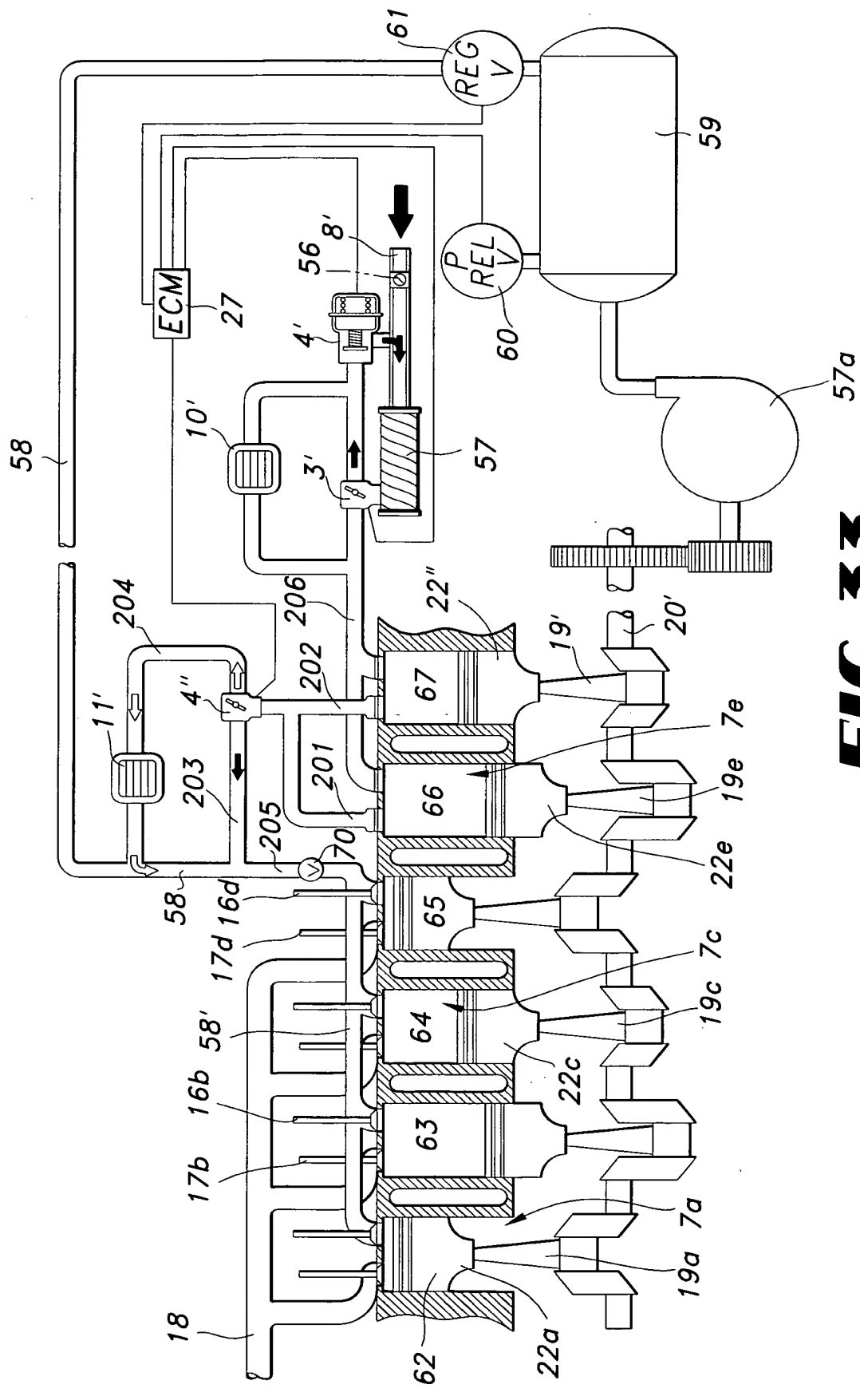


FIG 33

CLASS
DATE

08363103-052397

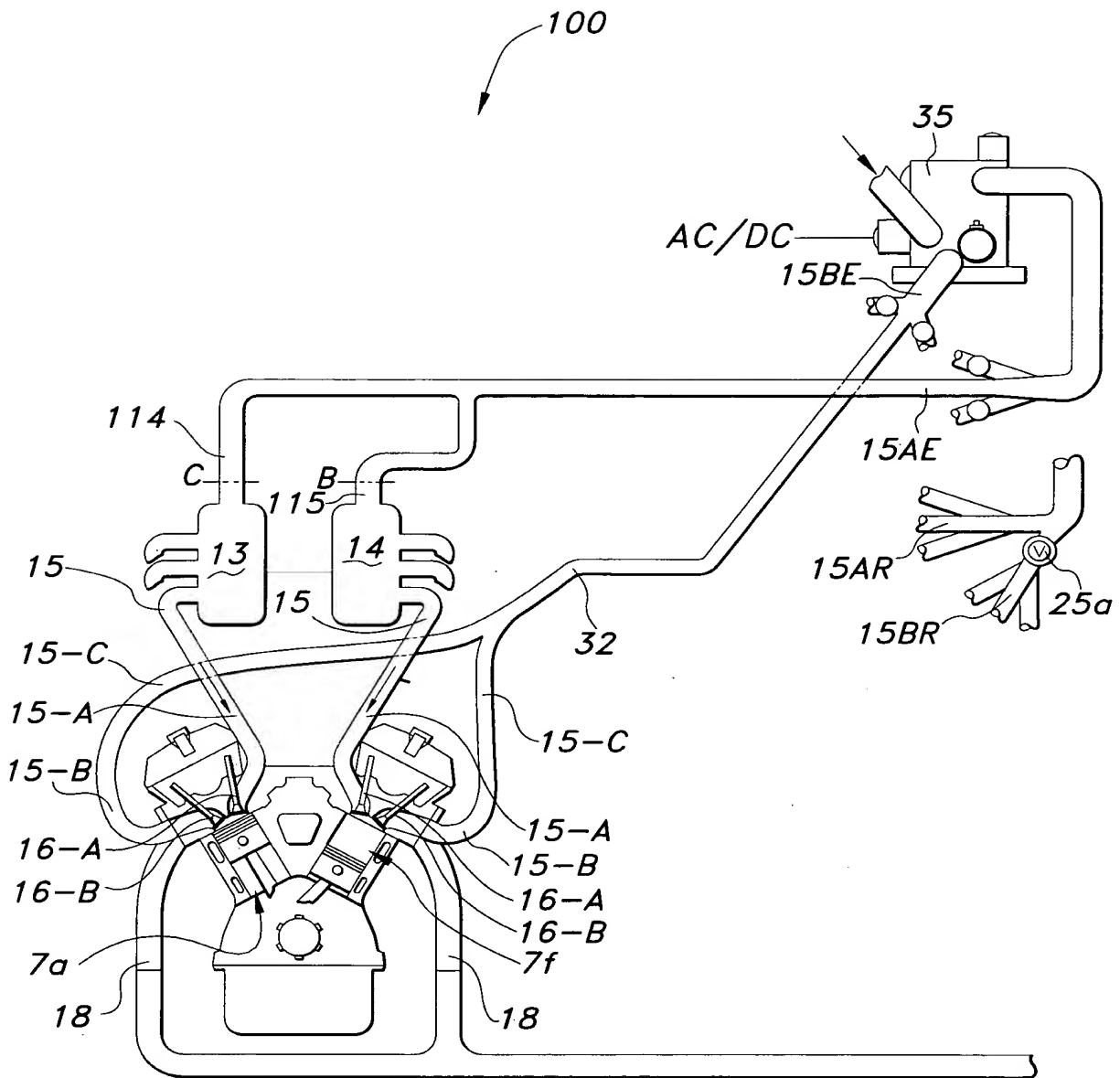


FIG 34

FIG 35

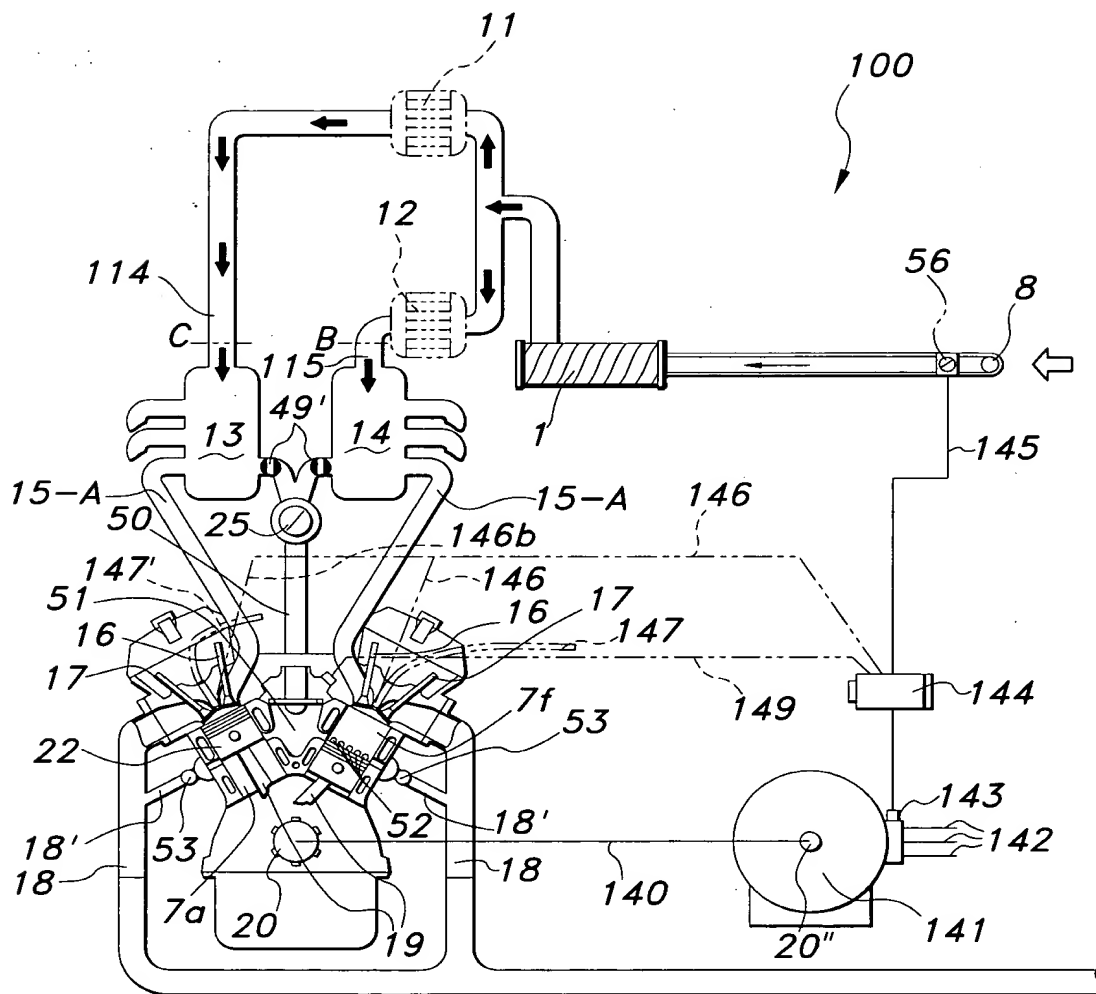


FIG 36

CANCELLED